

Recent progress, challenges and outlook for multidisciplinary structural optimization of aircraft and aerial vehicles

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Abstract

Designing an airframe is a complex process as it requires knowledge from multiple disciplines such as aerodynamics, structural mechanics, manufacturing, flight dynamics, which individually lead to very different optimal designs. Furthermore, the growing use of Carbon Fibre Reinforced Plastics (CFRP), while allowing for more design freedom, has at the same time increased the complexity of the structural designers job. This has sparked the development of Multidisciplinary Design Optimization (MDO), a framework aimed at integrating intelligence from multiple disciplines in one optimal design. Initially employed as a tool to coordinate the work of several design teams over months, MDO is now becoming an integrated software procedure which has evolved over the decades and has become a prominent tool in modern design of aerostructures.

A modern challenge in airframe design is the early use of MDO, motivated by a pressing industrial need for an increased level of detail at the beginning of the design process, to minimize late setbacks in product development. Originally employed only during preliminary design, MDO has recently being pushed into early evaluation of conceptual designs with the outlook of becoming established in the conceptual stage. Using MDO during conceptual design is a promising way to address the paradox of design. By improving each concept, evaluating whether it is capable of meeting the design requirements and computing the sensitivities of various performance measures with respect to a design change, MDO enables designers to gain valuable knowledge in a design phase, in which most of the design freedom is still available.

40 We hereby exhibit the contemporary trends of MDO with specific focus on composite
41 aircraft and aerial vehicles. We present the recent developments and current state-of-the-
42 art, describing the contemporary challenges and requirements for innovation that are in
43 the development process by academic and industrial researchers, as well as the challenges
44 designers face in further improving the MDO workflow. Within the European OptiMACS
45 project, we devised a novel holistic MDO approach to integrate a number of solutions to
46 challenges identified as industrial technological gaps. These include two-stage optimization
47 for layers of composites, addressing the presence of process-induced distortions and consid-
48 eration of advanced failure criteria, including refined local models in early design stages,
49 and seamlessly integrating software tools in the design process. The proposed methods are
50 integrated and tested for structural case studies and the obtained results show the potential
51 benefits of their integration into MDO tools.

52 *Keywords:* Multidisciplinary optimization, Aerostructures, Aerial vehicles, Manufacturing
53 informed optimization, Aircraft composite structures

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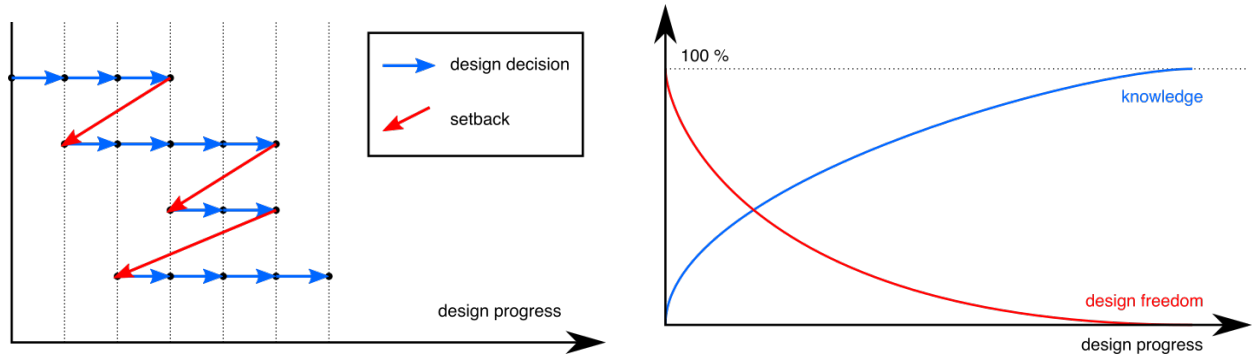
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82 **1. Introduction**

83 The design optimization of aeronautical structures for sizing of primary structural
84 components (wings, large portions of the fuselage) or even an entire aircraft is largely based
85 on Multidisciplinary Design Optimization (MDO). Due to modern aircraft structures being
86 largely made of advanced composite layered materials, in the design procedure the structural
87 design parameters that have to be determined during the MDO has radically increased. For
88 example, additional parameters to be determined as early as possible in the design pro-
89 cess include the exact layering design of the structure (i.e. number, sequence, thickness and
90 mechanical characteristics of each ply) and the manufacturing process to be followed for
91 each component. The ensemble of the design variables has to be simultaneously considered
92 and optimized vis-à-vis the adopted design criteria and constraints. In addition, composite
93 structures require sophisticated numerical models with consideration of advanced failure cri-
94 teria and description of the material at a mesoscale level, making the optimization process
95 a computationally demanding task. Hence, there is a genuine industrial need for develop-
96 ing advanced MDO procedures that are able to reliably provide the optimal design of the
97 composite structure under consideration within a rational amount of time.

98 As for any other complex product, the design of an aircraft structure starts with a
99 list of requirements and desired product characteristics. At the beginning of the design
100 process, engineers are free to make design assumptions, however at that stage they have
101 limited knowledge on how these decisions will address the target requirements. As the
102 design process advances, subsequent design decisions will always be constrained by the
103 previous ones, which may result in failure to satisfy the requirements, and the designers
104 will need to retrace their steps and redo the work (as illustrated in Fig. 1a). This problem is
105 known as the *design process paradox* [1], where in the initial design steps (when the design
106 freedom is maximal), there is a lack of information to guide the decision-making, while in
107 the final design stages, when there is sufficient knowledge, the design freedom is minimal, as
108 shown in Fig. 1b. Hence, considering the fact that the process of aircraft design is conducted
109 in three stages (conceptual, preliminary and detailed) and across a number of departments,

110 using computationally expensive numerical models, effective MDO procedures are required,
 111 preferably in early design stages, to gain more knowledge on structural performance ahead
 112 and allow for design flexibility without unnecessary repeated steps.



(a) Product development is characterized by steps forward and setbacks due to the violation of design requirements.

(b) The design paradox: as designers gain knowledge on how to design the product, they lose the freedom to modify the design.

Figure 1: Setbacks are normal in product development, but their opportunity cost increases as the design progresses, which leads to the design paradox.

113 A number of surveys have been published over the last two decades focusing on structural
 114 MDO with emphasis sometimes given towards mechanical analysis disciplines [3, 4], topology
 115 design considerations [5, 6] or optimization of stochastic parameters [7]. Literature reviews
 116 have also been provided towards computationally efficient schemes such as parallel archi-
 117 tectures [8] or methodologies involving the employment of response surface (surrogates) [9].
 118 Interest has also been rich considering optimization of geometrically complex architectures
 119 both regarding the mesoscale material design for composites [10, 11] and the macroscale
 120 geometric design at a component level [12–14].

121 Structural MDO has also been a topic of intense activity within the aeronautical industry.
 122 The need for lightweight and more efficient flying products has been steadily increasing over
 123 the last three decades given the rise in interest sourced from global travellers. Overarching
 124 survey reports have sporadically summarized the progress in aerospace MDO over time
 125 [15–18] with interest progressively shifting towards optimization of uncertain parameters
 126 [19, 20], as well as collaborative optimization [21, 22] specifically pertaining to complex
 127 structural areas [23]. Aeroelasticity is evidently a factor which can have radical impact on the
 128 structural design, with a few surveys [24–29] having been published regarding developments
 129 towards inclusion of aerodynamic and aeroelastic phenomena in the MDO framework. Before
 130 considering the comprehensive or even the preliminary aircraft structural design, MDO
 131 frameworks need to be able to determine optimal conceptual design choices for the vehicle.
 132 Hybrid electric architectures have been increasingly considered [30, 31], while blended wing
 133 body concepts [32] and wing morphing [33] are also design paths that need to be explored
 134 before reaching a decision. Inclusion of manufacturing constraints and uncertainty in the
 135 MDO process [34] is a topic which also received increasing interest, on which however few
 136 survey manuscripts have been published.

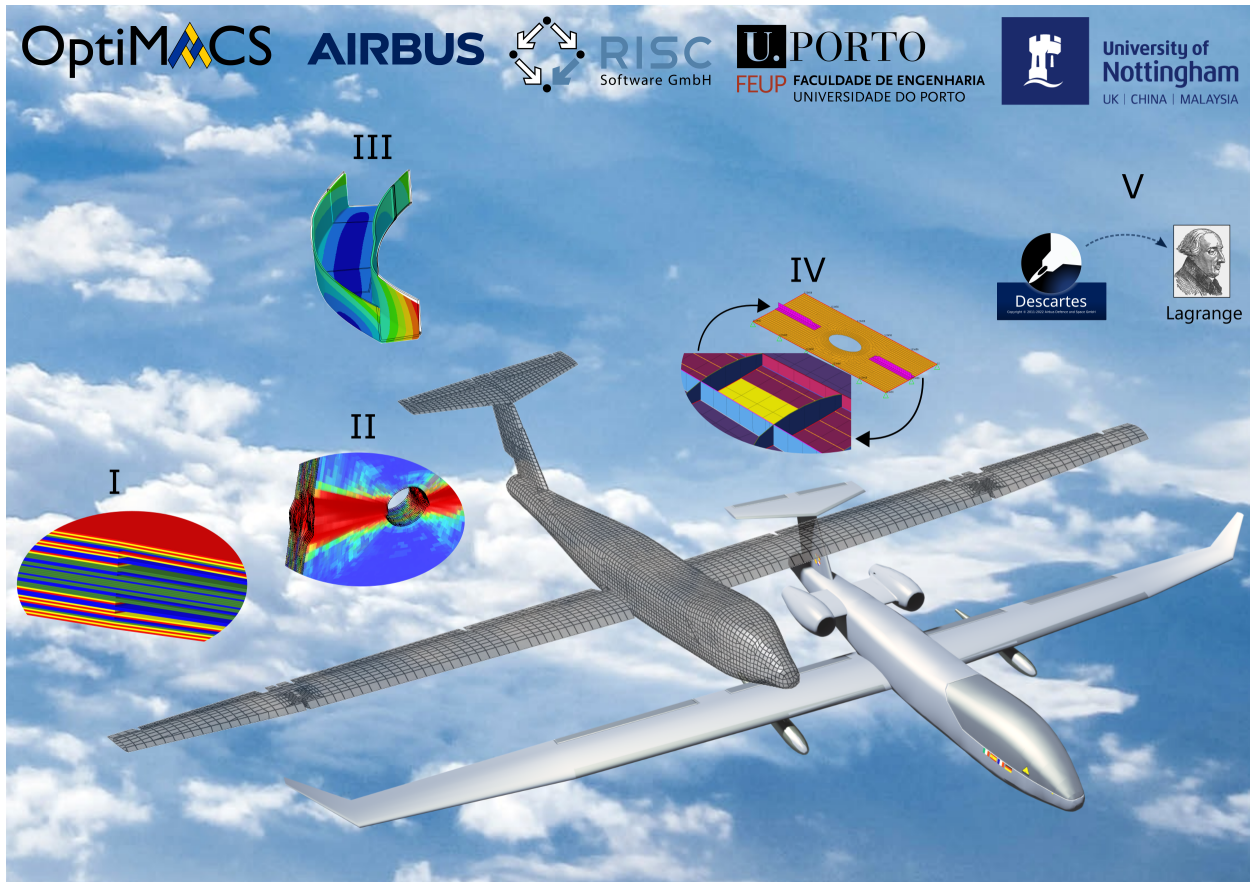


Figure 2: Contribution of each research topic explored in the Optimization of Multifunctional Aerospace Composite Structures (OptiMACS) project illustrated on OptiMALE [2], a Medium Altitude Long Endurance aircraft used as an academic demonstrator. In particular, these topics are: I) stacking sequence optimization; II) failure criteria and damage models; III) prediction of manufacturing distortions ; IV) global-local optimization; V) integration of software tools.

137 While MDO is to a certain extent still performed at a coarse resolution using low-fidelity
 138 models in a relatively manual process, the significant financial implications of the decisions
 139 made in the early design stages are putting an increased focus on achieving improved design
 140 accuracy, with a high degree of robustness. Specifically pertaining to the aerospace field,
 141 emphasis is therefore given to seamless integration of disciplines and a fine representation
 142 and resolution of geometric and material details wherever possible. New disciplines which
 143 were not explicitly accounted for until now (e.g. manufacturability and maintainability of a
 144 certain component) are also finding application within modern MDO frameworks.

145 A set of sophisticated software tools is currently employed within the European aerospace
 146 industry in order to perform structural optimization [35–37]. Such MDO platforms may
 147 have a number of design and performance optimization criteria implemented, with struc-
 148 tural weight, structural strength, aerodynamic and aeroelastic performance disciplines be-
 149 ing included amongst others. However, the number of computational and analysis stages
 150 performed through discrete software modules (i.e. the optimizer module, visualization and

151 manual post-processing modules, design verification modules and design drawing and export
152 modules) renders the modular data management a time consuming and counterproductive
153 task. Taking into consideration the several thousands of design variables required for a large
154 aerospace product, it becomes obvious that a genuine global industrial need exists for:

- 155 • Increasing the computational efficiency of the optimization models and algorithms
156 currently employed in the aerospace MDO platforms.
- 157 • Developing seamless procedures for facilitating modular data interchange during the
158 optimization process, and
- 159 • Extending the current set of adopted models and design criteria (also in view of the
160 recent advances in the fields of manufacturing processes and numerical characterization
161 of composite structures) in order to enhance the accuracy of the optimization process.

162 Motivated by these industrial needs, OptiMACS, a Marie-Curie research activity funded
163 by the European commission, was coordinated in order to deliver the most cutting-edge
164 research and training in the field of aerospace composite structures MDO through intense
165 training of five early-stage researchers. During the work on OptiMACS, the Airbus in-house
166 tools *Lagrange* and *Descartes* are used as a testbed to mature these new technologies:

- 167 • *Lagrange* is a multi-disciplinary structural optimization tool which has been contin-
168 uously developed since 1984 and applied to the design of various military and civil
169 aircraft. *Lagrange* consists of a general purpose finite element solver well suited to
170 the thin walled stiffened structures used in aerospace, optimization algorithms and
171 routines for evaluation of criteria models. Particular attention is paid to the modelling
172 of composite structures. The unique aspects of *Lagrange*, however, when compared to
173 commercial structural optimization codes, are the availability of the fully analytical
174 sensitivities of each system response to a given set of design variables and the inte-
175 gration of diverse linear aerodynamic analysis tools for aeroelastic and loads analysis,
176 including analytical sensitivity of aerodynamic and aeroelastic responses. This enables
177 highly efficient gradient based search of the design space for the optimum design. Sev-
178 eral optimization algorithms are implemented in the program to this end, each suited
179 to a specific type of optimization problem. These include both, first and second order
180 methods supporting a large number of design variables (approx. 10^5 - 10^6) and many
181 constraints (approx. 10^6 - 10^8). The automation of both load analysis and structural
182 sizing process is a key capability for the cost efficient development of high-performance
183 flying aircraft. See [35] for further information.
- 184 • *Descartes* is a flexible parametric geometry builder and automatically generates a
185 parametrised geometry model from an imported database in the Common Parametric
186 Aircraft Configuration Schema (CPACS) format [38] which has been developed by the
187 Deutsches Zentrum für Luft- und Raumfahrt - German Aerospace Center (DLR). The
188 CPACS XML data and design language provides all necessary characteristics of an
189 aircraft concept from one central database. The development of *Descartes* is based

190 on the TIVA Geometric Library (TIGL) , also from the DLR. *Descartes* can not
191 only import CPACS data, but it provides also the GUI to create a new aircraft model
192 from scratch based on engineering parameters. After the geometric model is set up,
193 *Descartes* can derive a finite element model as well as aerodynamic analysis models
194 and the coupling model (splining) between both. *Descartes* is intended for two major
195 purposes. First, to support the conceptual aircraft design process with numerical
196 analysis models, usually applied at the preliminary design phase, and second to enrich
197 the Multidisciplinary Design Optimization (MDO) process with the prospects of a
198 geometry kernel. Here, it can be used to morph the different analysis models during a
199 shape optimization process. See [39] for further details.

200 Concurrently with the need for developing the appropriate technologies, there is also a
201 need for exciting and motivating researchers regarding modern industrial technological gaps.
202 This is the prime incentive for developing this manuscript focusing on recent developments
203 pertaining MDO in Airbus, specifically within the frame of the OptiMACS project which
204 was a Marie-Curie research activity funded by the European Commission. Fig. 2 shows
205 schematically how each research work contributes to OptiMACS, where a novel holistic
206 MDO approach is developed to enable the exploitation of the complementary competences
207 of the researchers.

208 *Structure of the manuscript.* This paper is structured as follows: Sec. 2 gives a summary of
209 several important open questions and challenges that industry practice is facing when using
210 the currently available MDO procedures for optimization of aircraft structures with regard
211 to MDO architectures, optimization criteria and constraints. Sec. 3 presents the proposed
212 solutions developed within the OptiMACS project, addressing the practical challenges by
213 means of i) an advanced solution for layer design optimization; ii) advanced failure criteria
214 suitable for MDO processes; iii) consideration of Process Induced Distortions (PID) in the
215 manufacturing procedure; iv) a novel global-local MDO procedure for early design with up-
216 to-date local information in global models; and v) seamless integration of software tools and
217 automation in the design process. Sec. 4 presents practical examples of how the proposed
218 solutions have addressed the described practical challenges. Lastly, Sec. 5 concludes the
219 paper.

220 **2. Industrial challenges in aircraft conceptual design stage**

221 *2.1. Stacking sequence optimization of aircraft structures*

222 Modern aircraft are increasingly using lightweight structures made of CFRP. For load
223 carrying parts, the aeronautical industry relies mainly on continuous Unidirectional (UD)
224 carbon fibres due to their superior mechanical properties, in particular high stiffness and
225 strength, along the fibre direction compared to other forms of CFRP. The properties in
226 the transverse direction are mainly dominated by the plastic matrix of the composite and
227 thus such a UD material is highly orthotropic in nature. During production, UD plies, with
228 a fixed thickness dependent on the sourced materials, are stacked on top of each other to

229 form the required section of the part. This layered nature offers a huge design freedom. By
230 varying the orientation of the fibre angle and the order of the different plies, the mechanical
231 properties of the resulting laminate can be tailored to best suit the direction and scale of an
232 applied loading as well as to meet extensional or flexural stiffness requirements.

233 This advantage comes with the price of significantly increased complexity, as large struc-
234 tures such as wing covers can be made of hundreds of single plies. What is more, the
235 structural requirements of such large-scale components vary across their span and therefore
236 different regions or patches of these components must be modelled with stacks of differ-
237 ent thickness and composition while maintaining continuity or blending of the individual
238 plies. The thickness difference across different patches is visible during the Automatic Tape
239 Laying (ATL) manufacturing process employed for the component of Fig. 3. Additionally,
240 manufacturing requirements and the avoidance of negative mechanical behaviours further
241 constrain the task. These constraints influence either the stacking sequence of a laminate
242 itself, e.g. by requiring a symmetric or balanced lay-up, or the transition between laminates,
243 e.g. by restricting the maximum slope when ending multiple plies. A more complete list
244 of these rules and the reasoning behind them can be found in relevant works [40–42]. It is
245 worth noting that while some of these design rules are often relaxed in academic studies by
246 using more sophisticated analysis [43–45] and manufacturing methods [46–48] the industry
247 is following most of these rules to ensure robust processes and designs as well as simpli-
248 fying and thus de-risking certification of the aircraft by using known and well understood
249 principles.

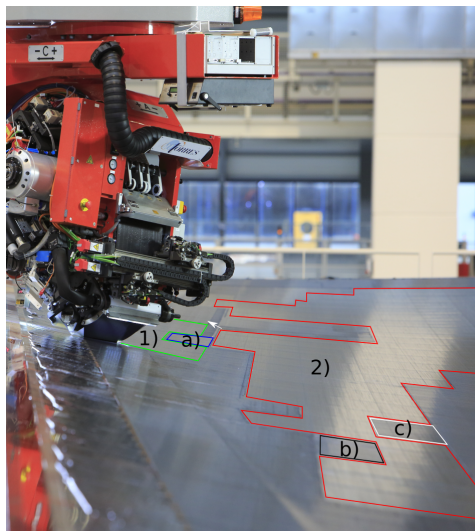


Figure 3: Visible skin patches formed by thickness differences across the span of the structure during the ATL manufacturing process of a wing cover.

250 The complexity in designing large CFRP structures therefore leads to surging develop-
251 ment costs and a higher chance of errors when using a traditional development process.
252 Therefore industry aims to mitigate this problem by applying optimization algorithms to
253 find feasible designs [36, 49–52].

254 The main challenge linked with the stacking sequence optimization of large-scale aerospace
255 structures is the mixed discrete and continuous nature of the problem. Structural constraints
256 such as strength, buckling, maximum displacements, etc. are formulated using continuous
257 quantities and depend on the solution of a FE-model which for large problems is significantly
258 expensive in terms of computational effort. On the other hand, design and manufacturing
259 rules concern discrete plies. The problem that arises is that different optimization algo-
260 rithms are better suited to tackle different parts of the problem. Heuristic algorithms can
261 be utilized to handle the discrete design variables and constraints however, such algorithms
262 require a lot of iterations especially when the design variables increase which is prohibitive
263 given the fact that physical constraints are linked with high computational expense. On
264 the other hand, gradient-based algorithms are well suited for the physical constraints of the
265 problem but do not perform well with its discrete characteristics.

266 This has led to two-stage approaches which first employ a gradient-based optimization
267 algorithm to get a continuous thickness and stiffness distribution of the structure followed by
268 a non gradient-based algorithm to handle the discrete requirements [53–55]. Unfortunately, a
269 gap is inevitably created between these two optimization stages which in order to be bridged
270 requires multiple iterations of the two-stage process, leading to a significant penalization in
271 terms of performance metrics of the aircraft and also added effort by multiple design teams
272 that have to repeat the process until a result fulfilling all structural and manufacturing
273 requirements is retrieved.

274 2.2. Failure criteria for composite materials

275 To identify feasible solutions when computing the optimum design of an airframe, the
276 corresponding optimization problem should account for different failure criteria among the
277 structural constraints. Failure in aerospace composite structures, in fact, is a complex
278 phenomenon which can occur due to different failure mechanisms: ply failure (where failure
279 can take place in the matrix, fibre and fibre-matrix interface), delamination, global buckling,
280 local buckling, crippling, column buckling, etc. The understanding of these phenomena gave
281 an essential contribution to describe the performance envelopes of aerospace structures.
282 Therefore, for the prediction of the onset of the mentioned failure mechanisms, as well as for
283 their propagation, a countless number of failure criteria and progressive damage modelling
284 approaches can be found in the literature. Recent reviews of these methodologies to address
285 failure in composite materials can be found in [56, 57].

286 For the prediction of ply damage onset, failure theories are usually classified in two
287 groups, by distinguishing theories that do not account for different failure modes, denoted
288 as *non-phenomenological* failure criteria, and failure theories that are able to identify the
289 different failure modes, denoted as *phenomenological* failure criteria [58, 59]. The first group
290 comprises criteria in which a failure envelope is typically defined by using a single mathemat-
291 ical expression, usually a polynomial form, which predicts failure by interpolating between
292 experimental data on simple (usually uniaxial) stress (or strain) states. Tsai-Wu and Tsai-
293 Hill are two common examples of non-phenomenological failure theories. Failure criteria of
294 the second family predict failure based on phenomenological considerations, by combining

295 different theories to model the specific failure modes. Among the available phenomenological
296 failure criteria, Hashin, Puck and LaRC failure theories can be highlighted.

297 The primary issue with including failure constraints directly in the structural optimiza-
298 tion problem is its resulting size. Indeed, in a typical application of structural optimization,
299 failure constraints may be enforced element-wise in the finite element model. However,
300 for detailed, high-fidelity structural models, this can lead to an optimization problem with
301 many thousands or millions of failure constraints, depending on the dimension of the model.
302 These constraints are costly to enforce because they can only be checked by completing the
303 structural analysis [60].

304 For the prediction of structural failure at the global level in MDO procedures, a commonly
305 used failure criterion for MDO is the maximum strain criterion, because of the multi-step
306 process used to determine the composite layup [61]. However, state-of-art approaches to
307 ply failure onset have achieved a high degree of accuracy, being able to represent several
308 relevant aspects of the failure process of laminated composites, e.g. the increase on apparent
309 shear strength when applying moderate values of transverse compression, or the detrimental
310 effect of the in-plane shear stresses in failure by fibre kinking. The most advanced set of
311 phenomenological failure criteria account for the effect of ply thickness, fibre misalignment
312 in compression, the effect of hydrostatic stresses and the effect of shear nonlinearity on fibre
313 kinking, and the in-situ strengths [62–65].

314 The industrial challenge linked with failure criteria is to use advanced phenomenological
315 failure theories to establish new laminate strength analysis models, suitable for optimization
316 purposes. A validation study of the predictions against experimental results will be required,
317 comparing the new model with the previous approaches in terms of reliability and efficiency.

318 Alongside the challenges in the deterministic methods, a recent effort has been made to
319 enhance the reliability of aerospace vehicles and decrease the chance of failure under potential
320 critical condition, by the development of non-deterministic approaches for optimization prob-
321 lems. In general two categories of uncertainty-based design methods can be distinguished:
322 reliability-based design optimization (Reliability-Based Design Optimization (RBDO)) and
323 robust design optimization (Robust Design Optimization (RDO)). RBDO allows to for-
324 mulate the probability of failure in the optimization problem. The aim of this approach
325 is to reduce the inherent conservatism of constant safety factors, which cannot weight the
326 potential uncertainties. However, for large-scale, highly non-linear, and non-convex prob-
327 lems, the deterministic MDO is already challenging and it naturally requires prohibitive
328 computational power to deal with uncertainties [16, 66]. For this reason, non-deterministic
329 approaches are not included in the industrial challenges of OptiMACS.

330 In the previous sections, the importance of increasing the level of accuracy in modern
331 MDO procedures from early stages of the design was highlighted. Global-local techniques can
332 represent a suitable answer to this challenge, thanks to their ability to capture the behaviour
333 of non-regular areas through local detailed models. Since this improved representation comes
334 at a reduced computational cost due to the local refinement, this approach is of great interest
335 for the aerospace industry. For instance, the structural sizing could benefit from detailed
336 models, where critical load cases can be correctly analysed.

337 Recently, few global-local procedures have been proposed to predict the global ply failure

338 and local skin-stiffener debonding of reinforced panels, while reducing the computational
339 time [56, 67, 68]. In particular, these methods proved to be reliable and efficient tools
340 to study localized nonlinearities, such as onset and evolution of damage, minimizing the
341 computational effort. For instance, a two-way coupling global-local approach, described in
342 [68], enabled to address delamination phenomena through a local analysis based on cohesive
343 elements and to ensure that the energy dissipated due to delamination evolution at the local
344 level was captured at the global level, with special attention to the exchange of information
345 between the global and local models. The validation of this method established a reliable
346 procedure of dissipated energy calculation for the global model due to delamination based
347 on the dissipated energy in the local model.

348 However, so far, only delamination has been properly addressed into these global-local
349 frameworks. For this reason, an additional challenge for OptiMACS is to tackle the integra-
350 tion of ply damage into an efficient global-local Finite Element (FE) approach.

351 *2.3. Prediction and compensation of distortions induced in composite structures by their* 352 *manufacturing processes*

353 The challenge in the manufacturing industry is to achieve a “First Time Right” approach
354 in order to increase product quality and reduce manufacturing costs and delays related to
355 manufacturing defects. In order to do so, in the manufacturing of composite structures it is
356 very important to monitor and minimize Process Induced Distortions (PID).

357 PID are the result of the combined effect of composite warpage and spring-in and can
358 be attributed to the residual stresses which are imposed within the structure during its
359 manufacture [69]. Almost every composite structure suffers from this manufacturing defect
360 to some degree.

361 Therefore, a tolerance range is set for each composite part within the structure to be
362 manufactured. This will not only increase the structural performance of the part due to the
363 reduction of the respective residual stresses, but will also ease the assembly process, reduce
364 the assembly time and costs. If the structure is outside of the tolerance range it is scrapped
365 and the tool which has produced the part has to be modified - if not completely redesigned
366 - to produce the desired geometry. In some cases, the design (geometry, materials, or layup
367 strategy) of the final product has to be reconsidered in order to reduce its shape distortions
368 after manufacturing.

369 Fig. 4 depicts what it is done currently in the industry to compensate for PID in the
370 design of new moulds.

371 The challenges to be addressed in the design process of moulds for composite structures
372 are :

- 373 • To increase material modelling accuracy (Step 2 & 4 in the design process as depicted
374 in Fig. 4)
- 375 – Investigation of advanced material models (viscoelastic vs elastic or modified
376 elastic models)

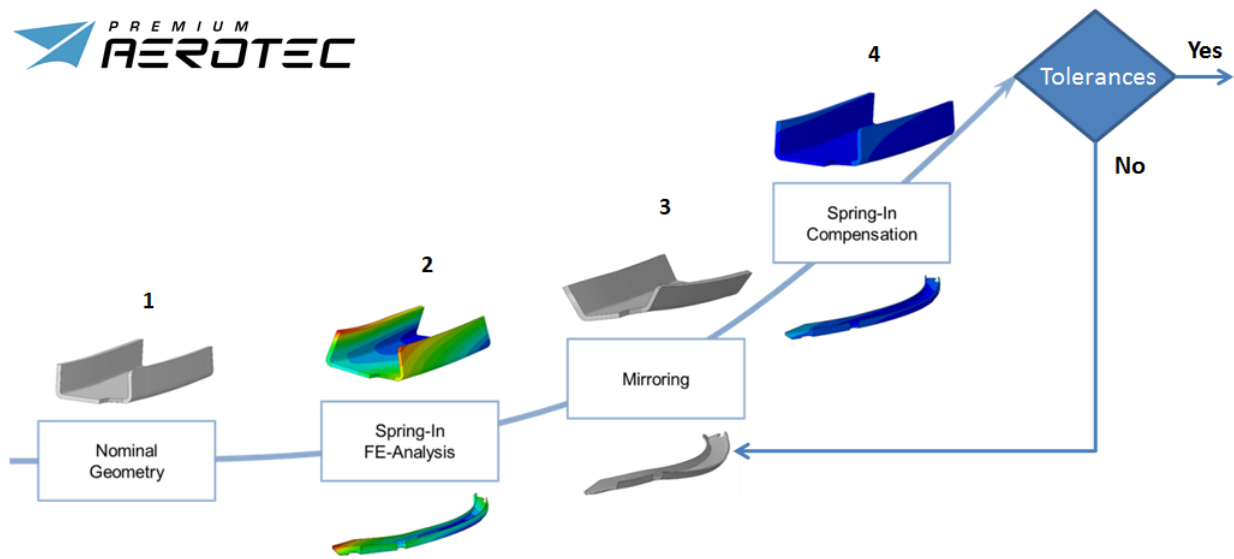


Figure 4: Design approach to produce the compensated mould geometry [70].

- 377 – Couple the material models with advanced functionalities (tool-part interaction,
 378 heat transfer analysis, chemical shrinkage strains calculation)
- 379 – Investigate different boundary conditions and their effect on simulation results.
- 380 • Automate the mirroring process for mould compensation (Step 3 in the design process
 381 as depicted in Fig. 4). The motivation here is to give the stress engineers all the tools
 382 they need to perform spring-in analysis without interfering with other departments
 383 (CAD, etc.) which is a time consuming and costly process especially in big companies.
- 384 • Experimentally validate the developed spring-in simulation framework. For the mate-
 385 rial system under investigation assess the simulation framework accuracy in academic
 386 and industrial parts. Identify sources of potential error to improve either the modelling
 387 or the manufacturing side of the structure.

388 2.4. Global-local multidisciplinary optimization of airframe structures

389 Traditionally, multidisciplinary structural optimization of aircraft employs coarse FE-
 390 models often combined with analytical post-processing to compute e.g. plate stability using
 391 stresses from the linear Finite Element Method (FEM). This implies that local areas with
 392 complex structures or load concentrations are idealized in a simplified manner which does
 393 not capture all effects. Hence this representation rarely provides enough information to
 394 analyse this area sufficiently during early design phases and it is hardly ever appropriate
 395 for sensitivity analyses. Frequently, the stiffness of these local areas is estimated by sim-
 396 ple methods and engineering judgement and the region is fixed during the optimization.
 397 However, if this manual step is not sufficiently accurate, designers may be forced to accept
 398 sub-optimal local solutions or require global changes at a later stage. Especially these late

399 changes in the global design lead to setbacks and are hard to recover. Therefore, a general
400 trend in the industry to include more details in preliminary design can be seen [36, 37].
401 This is especially true in the context of MDO where the fidelity level at the system level
402 is determined by the discipline with the lowest-fidelity model [71]. Additionally, this lack
403 of information limits the ability of designers to compare different concepts during the very
404 early stages, since the data regarding predicted performance is inaccurate [72].

405 While it becomes clear that it is necessary to include such details in early design stages,
406 the performance is crucial when analysing a complete aircraft or large components in a multi-
407 disciplinary optimization. This prohibits refinement of the full model and thus it makes sense
408 to employ so-called global-local techniques [73–80]. The non-regular local area is idealized
409 with a refined model which allows capture of all relevant effects and size this area. Reduction
410 methods such as [81] can be used to obtain accurate, but reduced stiffness and mass matrices
411 which will be used in the global model. The global model itself is rather coarse, but sufficient
412 to represent the overall stiffness and mass and allows the sizing of regular areas, while having
413 reasonable computational cost. At system level, only the global model is used for studies of
414 aeroelastic tailoring, flutter analysis, etc., while the local model might only be used during
415 the structural sizing and could even be skipped for loadcases known not to be design-driving
416 for this area. This allows avoidance of excessive computational cost, while capturing the
417 necessary information. A further challenge is to select an appropriate architecture for the
418 MDO process which is able to exchange and use the information relevant to the analysis such
419 as mass and stiffness properties, but also sensitivities with regards to the design variables
420 and the corresponding displacement field.

421 2.5. Integration

422 Evaluation of airframe design requires cooperation and transfer of information between
423 teams working on different stages of the design process. The multidisciplinary design team
424 add structural and aerodynamic details to an initial airframe design created by the concep-
425 tual design team. The performance of this more detailed design is then evaluated based on
426 the desired objectives and metrics. This is a process which is time-consuming and which
427 currently requires manual transfer of data between the different software packages used dur-
428 ing the process as shown in Fig. 5. This figure illustrates the workflow from the initial
429 airframe design in CPACS format [38] to model generation using *Descartes* [39] and finally
430 optimization using *Lagrange* [35]. The blue rectangles in the figure highlight where manual
431 transfer of data is currently required with the consequence that the current workflow to
432 evaluate the performance of a single airframe design may take one to two months.

433 Due to the manual nature of the data transfer interfaces, the length of time for airframe
434 evaluation is currently a barrier to the ability to perform any optimization for the overall
435 airframe design. Automation of these interfaces will make the use of MDO feasible for the
436 exploration of multiple airframe design variations.

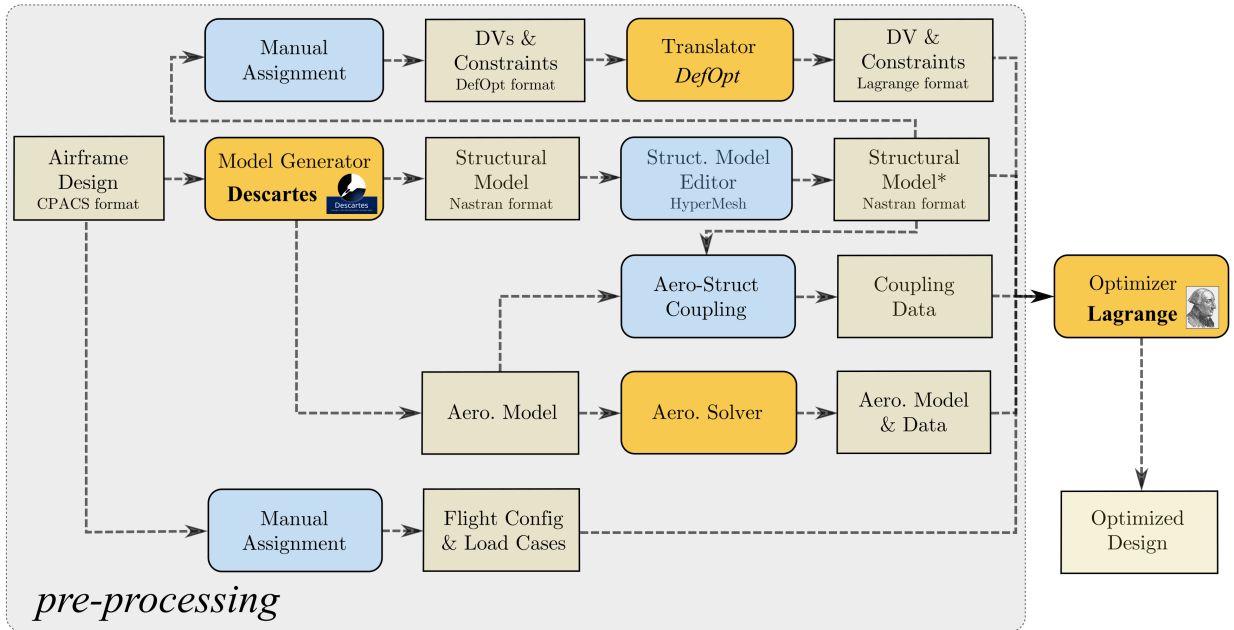


Figure 5: Current airframe design performance evaluation workflow. The airframe design (in CPACS data format) flows from the model generator (*Descartes*) and computation of input data (within the “pre-processing” box) to the optimizer (*Lagrange*) in order to obtain the final optimized design.

437 3. OptiMACS contributions

438 3.1. Stacking sequence optimization

439 As discussed in Sec. 2.1, two-stage optimization approaches offer the most potential for
440 successfully deriving discrete stacking sequences for aeronautical structures. In the frame-
441 work of the OptiMACS project, a two-stage optimization has also been adopted to perform
442 the optimization task. In contrast to other approaches employing lamination [53, 82, 83]
443 or polar [55, 84] parameters to model the thickness and stiffness of the structure in the
444 first, gradient-based stage of the optimization, the methodology adopted within *Lagrange*
445 uses generic stacks to model the properties of the structure. A generic stack is composed
446 of multiple generic layers whose exact orientation and stacking sequence is fixed during the
447 optimization. Therefore, the design variables employed during the optimization correspond
448 to the individual thickness of each generic layer, which can take any real positive value. In
449 the most simple case demonstrated in the optimization flowchart of Fig. 6, each generic stack
450 models one patch of the structure and only comprises 8 generic layers. In other words, the
451 generic stack used is $[45, -45, 90, 0]_s$. If symmetry of the laminated structure was also en-
452 forced in the optimization study, then this would lead to 4 design variables, while if balanced
453 laminates were additionally required, only 3 design variables per patch would be needed for
454 this simple generic stack. In practice, generic stacks with at least 16 generic layers should
455 be used in order to achieve an adequate representation of the stiffness design space. The
456 number of plies and stacking sequence of the generic stack need to be chosen so that the
457 resulting thickness and stiffness do not depend on the modelling decisions in the part. More

458 guidelines on these modelling decisions have already been discussed in previously published
 459 work by some of the authors [85], and it has been shown that a reduced number of design
 460 variables can adequately model even thick industrial-scale structures.

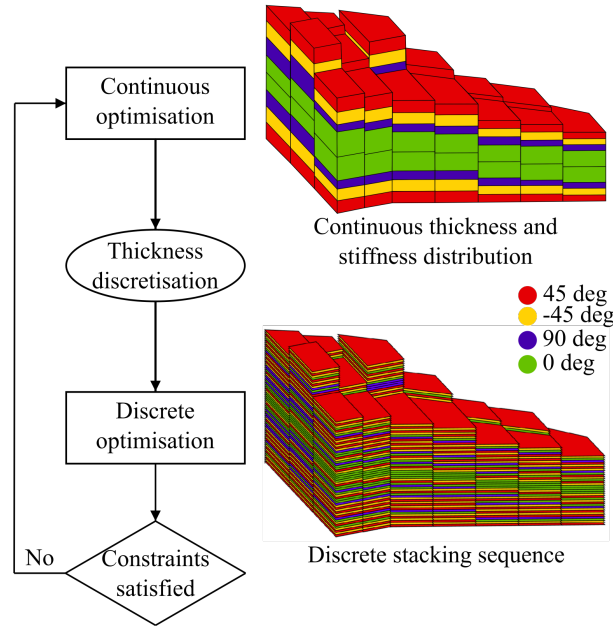


Figure 6: Flowchart of the two-stage stacking sequence optimization process.

461 The optimum continuous solution computed by the gradient-based optimization needs to
 462 be translated into a discrete stacking sequence which satisfies all the composite design and
 463 manufacturing rules. Since the thickness of the pre-impregnated tape that will be used for
 464 manufacturing is known, the entire thickness of each generic stack is rounded-up to either
 465 the nearest integer number of plies or the nearest even number of plies. This discrete number
 466 of layers for each patch remains constant during the discrete optimization.

467 The second stage of the optimization involves mathematical programming and solving a
 468 Mixed Integer Linear Programming (MILP) formulation of the stacking sequence optimiza-
 469 tion subject to any composite rule, aiming to match the stiffness characteristics derived in the
 470 first stage as accurately as possible. Two equivalent formulations of the stacking sequence
 471 optimization problem subject to various composite design and manufacturing rules have
 472 been derived [42]. These MILP formulations offer more design freedom than most available
 473 blending representations [41, 86, 87] and also achieve a more robust convergence towards
 474 the global optimum compared to heuristics. A decomposition technique which renders the
 475 optimization algorithm a heuristic one has also been developed to assist with the discovery
 476 of good local optima in a much shorter time frame for industrial-scale problems. The quality
 477 of these local optima may be adequate enough, rendering the search for a global minimum
 478 to the problem unnecessary. If this is not the case, the local optima can be used to initialize
 479 the non-decomposed formulation of the problem to enable faster overall convergence of the

480 optimization.

481 The proposed two-stage optimization can consistently lead to discrete stacking sequences
482 which satisfy all required structural constraints in only one pass of the two-stage process.
483 This is mainly due to the fact that the design space of generic stacks allows for more compos-
484 ite rules to be implemented, since it is a direct representation of the sequence characteristics
485 of the stacks. Moreover, the formulation of blending, in particular, is exact, compared to
486 other approximate formulations when using other modelling approaches, such as lamination
487 [54] and polar [88] parameters. As a result, the gap which is inevitably formed between
488 the two optimization stages is bridged and the entire design process is simplified and ac-
489 celerated. Moreover, by bridging this gap, the current methodology is able to achieve a
490 lower structural mass when applied to a benchmark problem [85] compared to other studies
491 sharing equivalent design criteria.

492 *3.2. Integration of failure criteria in the MDO process*

493 In Sec. 2.2, the challenges linked with the integration of failure constraints in large
494 structural optimization problems were described. Additionally, another objective of this
495 research work is to tackle the integration of ply damage and delamination into efficient
496 global-local procedures. In the next sections, the methodologies implemented to address
497 these research topics are described in detail.

498 *3.2.1. Failure constraints in strain space for global MDO problems*

499 Strength constraints suitable for MDO are required to outline safe failure envelopes for
500 different loading conditions that do not compromise the efficiency of the optimization pro-
501 cess. Another desirable aspect regarding failure criteria for MDO is their formulation in
502 strain space, mainly because they can benefit from invariant laminate failure predictions
503 with respect to ply orientations, simplifying the design of composite laminates. In fact, the
504 failure envelope for a given ply angle is fixed in strain space independently of the orientation
505 of the other plies in the laminate, unlike ply failure envelopes in stress space [89]. This
506 means that these envelopes, as well as the inner failure envelope in strain space of a multi-
507 directional laminate, can be viewed as material properties [90]. Failure envelopes in strain
508 space, therefore, enable an invariant description of failure with respect to the ply orientation,
509 which is essential for a continuous optimization with lamination parameters [91, 92], and to
510 quickly compare against the maximum allowables obtained experimentally.

511 The challenge of this research work was to develop a laminate strength analysis method
512 using an advanced phenomenological failure theory. To tackle this challenge, an extended
513 failure prediction approach, based on a recently introduced concept called omni strain failure
514 envelope [90, 93], was developed in order to address laminate failure under general 3D stress
515 states and to identify critical failure modes [94]. A omni strain envelope is an envelope
516 obtained by superposing failure envelopes for all possible ply orientation in strain space and
517 extracting the inner design space. In fact, in strain space it is possible to superimpose the
518 failure envelopes for the different ply orientations and compute a laminate failure envelope.
519 This theory was originally proposed using the Tsai-Wu failure criterion applied at the ply
520 level. With this approach, all laminate data can be displayed on one graph in strain space,

521 realizing a very concise display of the strength of a given composite material. Furthermore,
522 it is a very practical tool, enabling a fast selection of the stacking sequence according to the
523 required mechanical properties, since it covers all the possible ply orientations.

524 Because the omni First-ply failure (FPF) envelopes represent the most conservative de-
525 sign solution, where all the plies remain undamaged, Tsai and Melo [93] proposed an ex-
526 tended version of this criterion, to define and predict the continued load-carrying capability
527 of any laminate, after damage initiation. They introduced the omni Last-ply failure (LPF)
528 envelope, which is an extension of the concept of omni FPF envelope to ultimate failure.
529 The construction of these envelopes follows the same procedure as described before, but
530 with degraded ply properties, based on a matrix degradation factor and micro-mechanics
531 relations. Moreover, Tsai and Melo [93] observed that, for all CFRP laminates, the inner
532 LPF envelope is controlled by the 0° and 90° plies loaded along the respective fibre direction.

533 Based on these observations, a further simplification of the failure analysis was performed
534 introducing the unit circle failure envelopes for CFRPs in normalized principal strain space,
535 which rely on just two strength properties: the longitudinal tensile and compressive strains-
536 to-failure. Comparing the omni strain LPF envelope and the unit circle failure envelope of
537 the same material, the unit circle envelope is inscribed in the omni LPF envelope. Although
538 the failure predictions related with this criterion are intentionally conservative, this theory is
539 extremely useful due to its simplicity. In particular, by requiring only the strains-to-failure
540 of a 0° coupon measured in tension and in compression instead of complete characterization
541 of the ply properties, not only the failure predictions, but also the material characterization
542 can be substantially simplified.

543 By exploiting the fully 3D description of failure provided by the invariant-based theory
544 proposed in [65], omni strain failure envelopes can be extended to omni strain failure surfaces
545 by finding the controlling plies in the 3D principal strain space. Indeed, with this extension,
546 the resulting design space can predict failure under complex 3D stress states of any lami-
547 nate, independently of lay-up or stacking sequence, and address, for instance, the design of
548 bolted joints or thick composite laminates, where through-thickness stress states cannot be
549 neglected. Furthermore, in this case, the envelopes allow the identification of the critical
550 failure modes for each controlling ply, which cannot be investigated with the Tsai-Wu based
551 omni strain envelopes. An illustration of this extension of omni failure criteria is provided
552 in Fig. 7, while a detailed presentation of this work is provided in [94].

553 It is also important to note that, for typical CFRP laminates, such as aerospace industry-
554 standard “quad” laminates, characterized by different percentage of 0° , $\pm 45^\circ$ and 90° plies
555 [95], omni LPF and laminate LPF envelopes (the latter obtained from superposing in strain
556 space only the envelopes of the ply orientations contained in the selected laminate) will
557 lead to the same laminate failure predictions. This is justified by the presence of the [0]
558 and [90] plies in these laminates, which will govern LPF according to both approaches.
559 Therefore, for all CFRP “quad” laminates, the omni LPF envelopes ensure the same degree
560 of conservatism as the laminate LPF envelopes, but without the need to recompute the
561 failure envelope every time the layup changes. However, when tackling LPF of angle-ply
562 or double angle-ply (double-double [95]) laminates, omni LPF envelopes will have a certain
563 degree of conservatism that will depend on the ply angles.

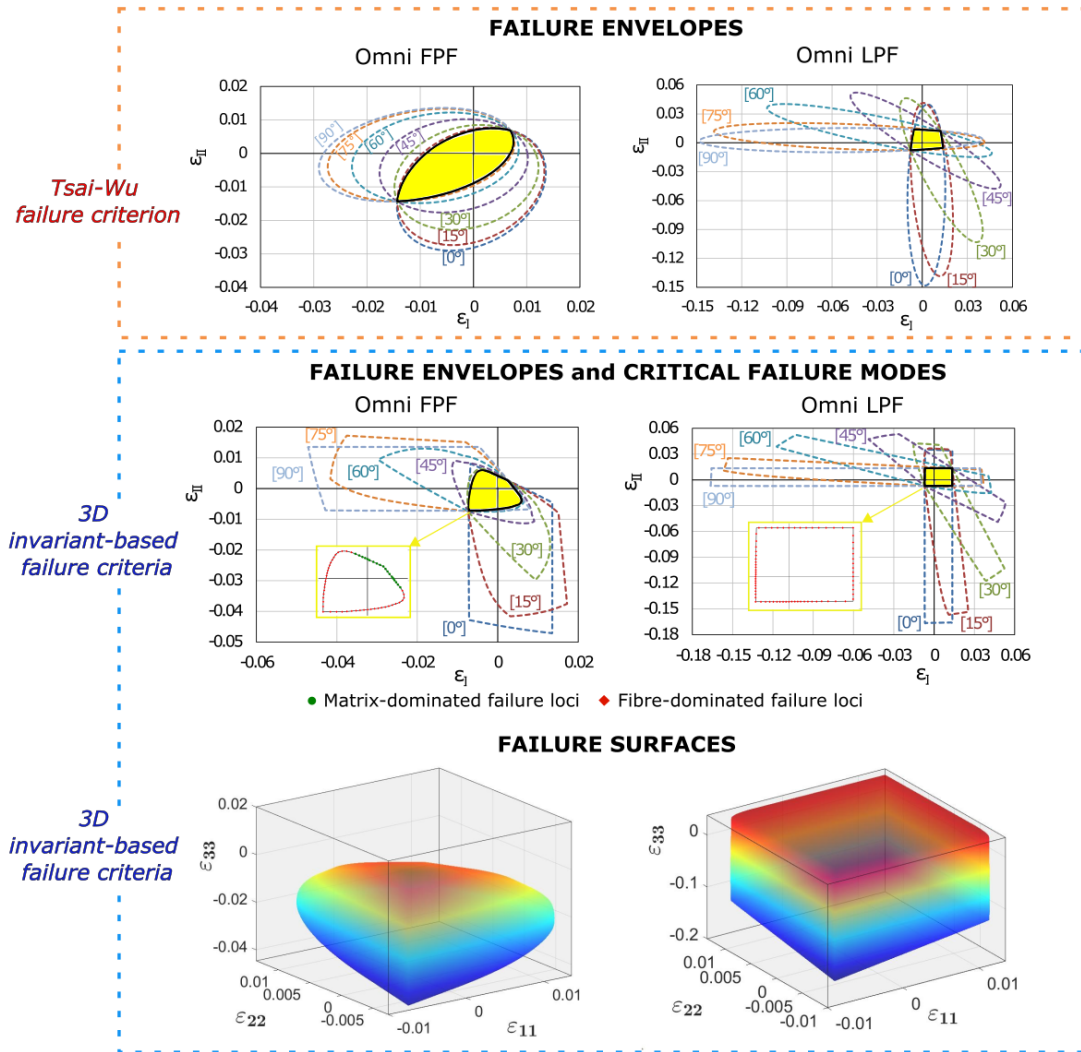


Figure 7: Omni strain failure envelopes and surfaces.

564 *3.2.2. Detailed progressive failure analysis for local models*

565 Global-local analysis is typically performed at the subcomponent level to address critical
 566 phenomena with detailed models and to translate the obtained results into the global scale.
 567 In this framework, failure criteria can be firstly implemented in the global level for “hot
 568 spot” identification to provide a first indication of the critical locations where detailed local
 569 analysis should be performed.

570 The implementation of failure criteria in a FE software for hot-spot failure analysis has
 571 been recently proposed in [96–98]. Exploiting the indications obtained from a hot-spot failure
 572 analysis, it is possible to increase the modelling resolution only where required, enabling
 573 a more efficient and reliable global-local analysis, especially when addressing large-scale
 574 composite structures. Molker et al. [96] proposed the implementation of LaRC05 for the
 575 prediction of failure initiation and critical failure modes of laminated composite structures,

576 while a novel LaRC05-based failure theory was implemented to address damage onset of
577 Non-Crimp Fabric (NCF) reinforced composites [98]. These implementations were done in
578 the commercial FE code Abaqus/Standard, by means of a user defined subroutine UVARM
579 creating element output variables at each integration point and each time increment.

580 As part of the OptiMACS project, the 3D invariant-based failure theory [65] is used for
581 the prediction of FPF of laminated composite structures. With this aim, the formulation
582 of this set of criteria was implemented in a post-processing Python script, compatible with
583 Abaqus/Standard and Abaqus/Explicit, to generate new element output fields, by using the
584 full stress tensor of each element and computing the failure index for each of the failure
585 modes tackled by the criteria. Because of the fully 3D nature of the implemented failure
586 theory, this approach allows the identification of “hot-spots” and the corresponding failure
587 mechanisms for damage initiation, creating different output variables to predict fibre and
588 matrix failure, under tension and compression. On the other hand, when the aim is to
589 address LPF at laminate level, a model with an equivalent single layer discretization and
590 a laminate failure criterion should be used, which is particularly interesting in large-scale
591 structural models. With this purpose, an additional post-processing script is in place to
592 compute and show the failure index obtained with unit circle failure theory [93], by means
593 of an element output variable.

594 After identifying the “hot-spots” for the onset of ply or laminate failure, a detailed
595 damage model can be employed to predict ultimate strength of the most critical areas, while
596 representing all damage modes and their interactions. The damage modes taking place in
597 composite materials evolve in various combinations that depend, among other factors, on
598 the stacking sequence and ply thickness. Some combinations of damage may reduce local
599 stress concentrations, while others may cause structural collapse. This is the reason why it is
600 crucial to have a model that is able to predict damage initiation and propagation accurately
601 [99].

602 Among the different scales of idealization of the damage process, which may span from
603 molecular dynamics to structural mechanics, Fibre-reinforced polymers (FRPs) are most
604 commonly represented at the mesoscale, due to the flexibility it provides in representing in
605 detail the initiation and propagation of the different failure modes observed in FRPs within a
606 reasonable computational effort. Mesoscale numerical models have been recently developed
607 to represent the onset and broadening of the intralaminar damage modes (e.g., transverse
608 matrix cracking and fibre failure) and use cohesive zone models to capture delamination
609 between ply interfaces [100, 101].

610 The methodology introduced in this work consists of a composite material model pro-
611 posed in the literature [101], representing the quasi-brittle behaviour of composite structures.
612 It is extended to account for the effect of general 3D stress states in the initiation and broad-
613 ening of fibre kinking using the 3D invariant-based failure theory, as described in [65]. The
614 invariant-based failure criteria are coupled with a smeared crack model for transverse crack-
615 ing and continuum damage mechanics models for fibre-dominated damage, which together
616 account for the kinematics of matrix cracking and fibre tensile or compressive fracture dur-
617 ing damage propagation. Furthermore, to predict delamination, cohesive elements are used
618 at the interfaces between layers with different orientation. It should be noted that, to use

619 cohesive zone models properly, a minimum number of elements within the cohesive zone is
620 required. A quantitative study on this topic can be found in [102], where a procedure to use
621 coarser FE meshes is described, identifying a minimum of 3 elements within the length of
622 the cohesive zone to predict delamination growth without losing accuracy in the results.

623 *3.3. Consideration of PID in the manufacturing of composite structures*

624 To address the industrial challenges presented in 2.3 regarding the presence of Process
625 Induced Distortions (PID) in the manufacturing of composite structures the OptiMACS
626 project focused on the investigation of two material models regarding their ability to accu-
627 rately predict the shape of the manufactured part. With the use of these material models a
628 simulation framework was developed, able to take into account the majority of the factors re-
629 ported in the literature to affect PID such as resin chemical shrinkage, tool part interaction,
630 temperature gradients, stress relaxation, etc. [103–108]. The validation of the simulation
631 framework for the material system studied was done by experimental investigations in the
632 laboratory as well as with the study of an industrial size composite test frame.

633 More specifically after obtaining the certified stacking sequence of the composite struc-
634 ture, being a product of MDO analysis, the job of the manufacturing engineer is to design
635 the mould and the tools needed to manufacture the part according to its quality criteria,
636 as well as determine the manufacturing process suitable to produce the part (resin infusion,
637 resin transfer moulding, etc.) within the given budget and time. Because of the limited
638 design freedom in the late design phase of composite structures, MDO is rarely employed to
639 address the manufacturing defects arising from the manufacturing process selected. These
640 manufacturing defects include, but are not limited to delamination, fibre wrinkling, fibre re-
641 orientation, fibre pull-out, increased void content, etc. and usually a multi-physics analysis
642 approach coupled with experimental investigations and manufacturing experience is used to
643 address them and optimize the manufacturing process [109–112].

644 PID is a significant problem encountered in the manufacturing process of composite
645 structures irrespective of the manufacturing method used. To counteract this manufacturing
646 defect the manufacturing engineer simulates the three dimensional shape distortions of the
647 part and mirrors the distortions to the mould by reversing the calculated part distortions
648 in the opposite direction (Fig. 4). This approach is referred to in the industry as the mould
649 compensation approach since the shape of the mould is different from the shape of the part
650 that is going to be manufactured from it. The final product after demoulding will distort and
651 if the calculations of the manufacturing engineer are correct, it will be close to its nominal
652 geometry (Fig. 8).

653 The material model employed by the manufacturing engineer in this process is very
654 important to accurately predict shape distortions. Three types of material modes are used
655 with increasing complexity and accuracy: elastic, modified elastic and viscoelastic material
656 models. The elastic material models focus on the development of the shape distortions of the
657 structure during the cool down phase of the curing cycle, when the structure has attained its
658 final degree of cure, and consider the material as elastic during this phase. On the other hand,
659 the modified elastic models separate the curing history into a number of segments to which
660 they assign an elastic modulus, in order to calculate the residual stresses and distortions

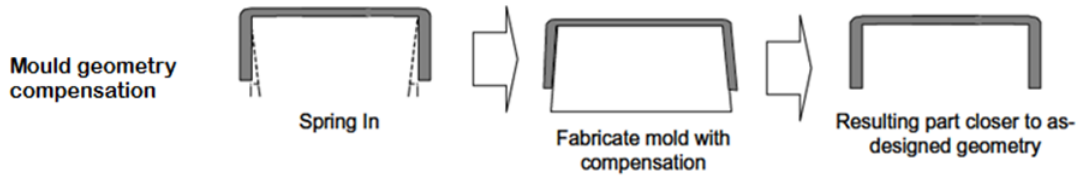


Figure 8: Mould compensation approach [113]. The spring-in angle of the part is known a priori either through manufacturing experience or simulation. Therefore, the mould of the part is compensated accordingly (usually by mirroring the expected distortion of the part to the opposite direction) in order for the resulting part after demoulding to be closer to the as-designed (nominal) geometry.

661 of the structure. The viscoelastic material models are time depended models able to take
 662 into account the stress relaxation of the material during the cure and are regarded by the
 663 literature as more accurate compared to the elastic or modified elastic models or the use of
 664 analytical equations [114–116].

665 In the context of the OptiMACS project the Cure Hardening Instantaneous Linear Elastic
 666 (CHILE) material model was investigated. The CHILE material model is a simple, robust
 667 and fast model used by the academia as well as by industry to calculate PID of composite
 668 structures. However, since it is not a time dependent model it cannot take into account
 669 stress relaxation which occurs during the curing. The second material model investigated
 670 in the context of the OptiMACS project is the linear viscoelastic model proposed by Poon
 671 and Ahmad [117].

672 To exploit the full capabilities of the material models these were coupled with functions
 673 that calculate at each time step the resin chemical contraction, glass transition temperature,
 674 resin coefficient of thermal expansion, resin instantaneous fibre volume fraction and Pois-
 675 son’s ratio before employing the ply homogenization equations proposed by Bogetti [118]
 676 as depicted for the case of UMAT subroutine in Fig. 9 employed for the chemo-mechanical
 677 spring-in analysis.

678 To further increase the accuracy of the simulation framework developed, tool part in-
 679 teraction was studied in comparison to free-standing and fixed boundary conditions. Free-
 680 standing boundary conditions imply that the 3-2-1 principle is used to suppress rigid body
 681 motion of the part during the curing. At the fixed boundary condition the Degrees of Free-
 682 dom (DOFs) 1,2 and 3 of the part are set equal to zero and as a result of the part not being
 683 able to move during the curing cycle.

684 Regarding the study of tool part interaction, a Coulomb friction approach was used in
 685 the context of the OptiMACS project. In the normal direction of the contact, “hard” contact
 686 was assumed meaning that the tool and the part could not penetrate each other. In the
 687 tangential direction a cure dependent Coefficient of Friction (CoF) was assumed from the
 688 gelation point to cool down. Instead of performing an experimental investigation to assess
 689 the evolution of coefficient of friction from the gelation point to cool down for the material
 690 system under investigation, a linear relationship was adopted from the literature [119].

691 The motivation for the development of a robust friction model to describe the forces that
 692 are transmitted from the mould to the part during the manufacturing cycle is to substitute

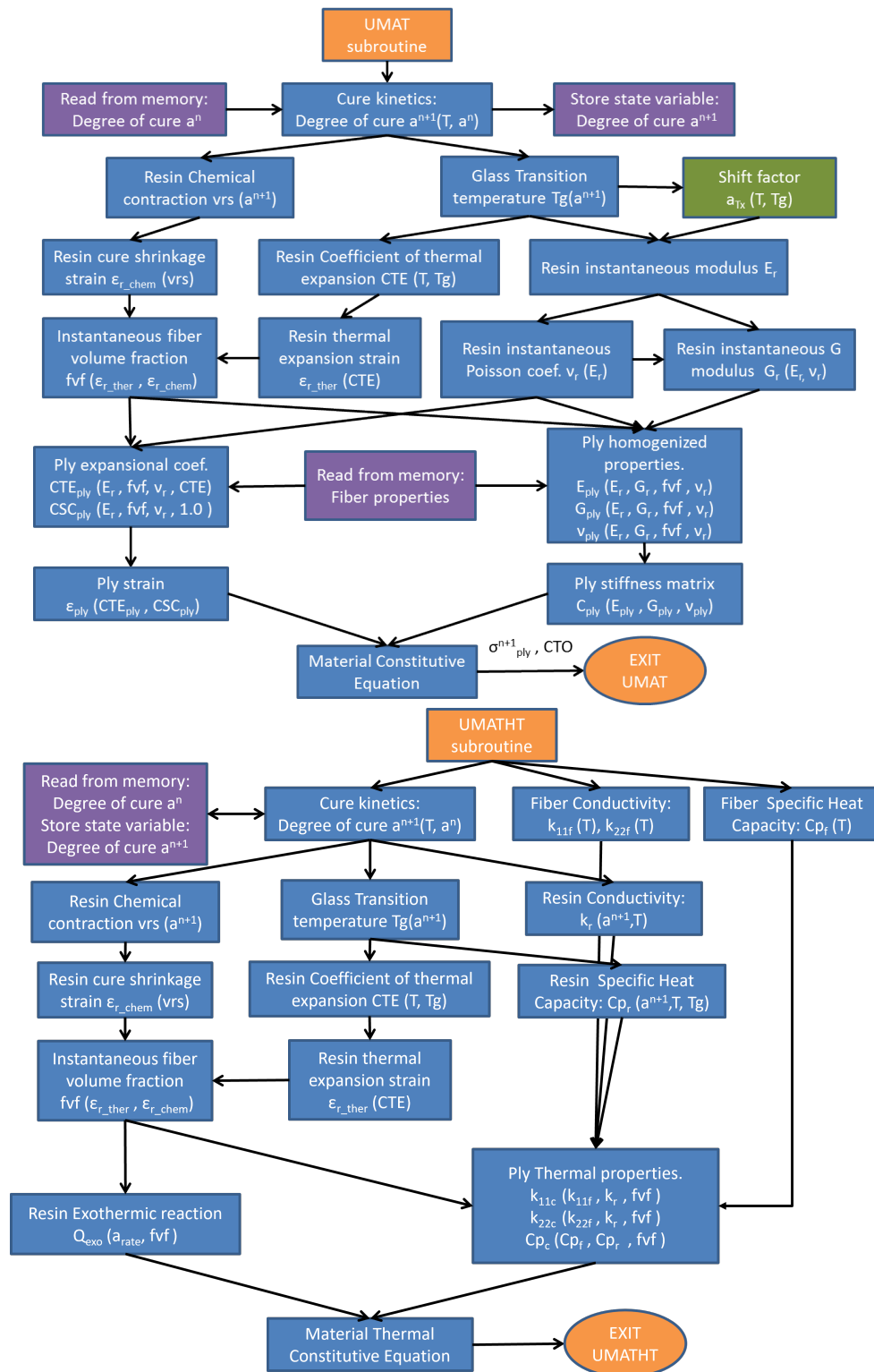


Figure 9: Diagram of the material modelling architecture (UMAT top used in the chemo-mechanical analysis, UMATHT bottom used in the thermal analysis). Blue colour indicates variable calculation, purple access of memory, orange start/finish of the calculation process whereas the green box is employed only by the viscoelastic material model [70].

693 the INVAR tools, which are now used in the aerospace industry to manufacture structural
694 parts because of their low Coefficient of Thermal Expansion (CTE) in accordance with the
695 CTE of composite structures, with cheap alloys such as steel or aluminium. However, this
696 requires experimental investigations for the material system and manufacturing process of
697 interest, in order to determine the parameters of the friction model. Furthermore, the draw-
698 back of adding contact behaviour in the simulation process is that it increases the complexity
699 of the simulation making it computationally demanding especially for large industrial parts.

700 In the context of the OptiMACS project, heat transfer analysis between the tool and the
701 part was also investigated with the aim to identify significant temperature gradients across
702 the part that could lead to property gradients in the part affecting its PID and dimensional
703 stability. Thus, the already developed chemo-mechanical simulation approach was extended
704 to a thermo-chemo-mechanical one. A subroutine (UMATHHT) was developed in ABAQUS
705 to take into account the exothermic heat reaction of the resin during cure and calculate the
706 effective lamina thermal properties, namely the effective lamina conductivity and specific
707 heat capacity as depicted in Fig. 9. With the use of this subroutine, the calculation of the
708 temperature gradients across the part during the manufacturing cycle can be made. The
709 challenge in the heat transfer analysis is to determine the heat transfer coefficient between
710 the part and the tool which is a function of many variables (turbulence of the air in the oven,
711 flow medium, pressure, temperature, etc.). In our case, two heat transfer coefficients were
712 employed, one for the mould side and one to simulate the heat flow from the vacuum bag
713 side of the part. The heat transfer coefficient values used in the simulation were supplied
714 by Premium AEROTEC GmbH which measured the heat transfer coefficient in one of its
715 ovens.

716 In the field of automation of the mirroring process for mould compensation the typical
717 process would be to send to the design department the results of the first FE spring-in
718 analysis in order to produce the updated mould surface. This surface is used usually for a
719 second analysis in order to verify that the final product lies inside the predefined tolerance
720 range before manufacturing the mould. In order to avoid the interaction between different
721 departments which usually result in delays in the design process, three scripts were developed
722 in the OptiMACS project to automate the mirroring process and produce the final mould
723 surface, reducing the relevant design costs. One is used to mirror the 3D distortion field
724 of the first spring-in analysis, the second one to translate and smooth the mesh of the
725 untrimmed area (yellow as depicted in Fig. 10) to fit the trimmed element group (green as
726 depicted in Fig. 10). This step is necessary because the trimming operation releases stresses
727 and the subtraction of an element group from the model affects its distortion field. Finally,
728 the last script is used to create CAD surfaces from the updated mesh geometry. Fig. 10
729 depicts the result of the application of the scripts in the mirroring process of the distortions
730 of a U-section which is part of a composite frame (distortions multiplied by a factor of five
731 for visualization purposes).

732 Finally, regarding the validation of the developed simulation framework, composite L-
733 shape specimens were manufactured in the laboratory and measured with a Coordinate-
734 Measuring Machine (CMM). Their spring-in angle was then compared with the simulation
735 predictions of the two material models. Three moulds were manufactured from steel, INVAR

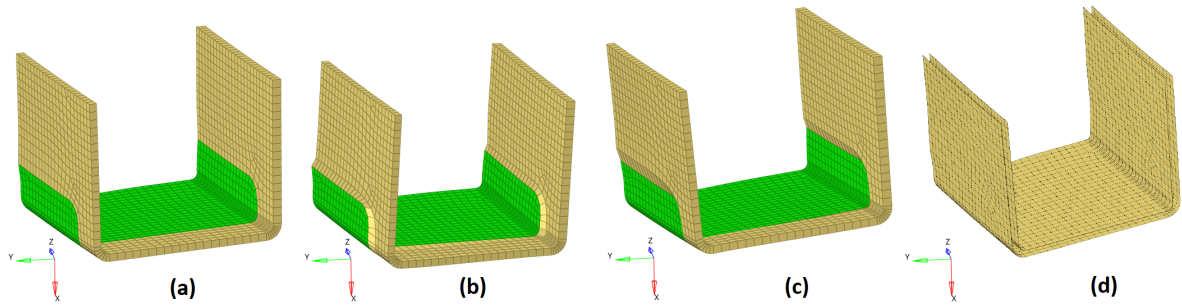


Figure 10: The steps employed to automate the mirroring process of the distortions of a spring-in analysis. a)Nominal geometry b)Result of the first spring-in analysis c) Mirroring of the distortions to the opposite direction c) Smoothing of mesh and creation of CAD surfaces (inner and outer).

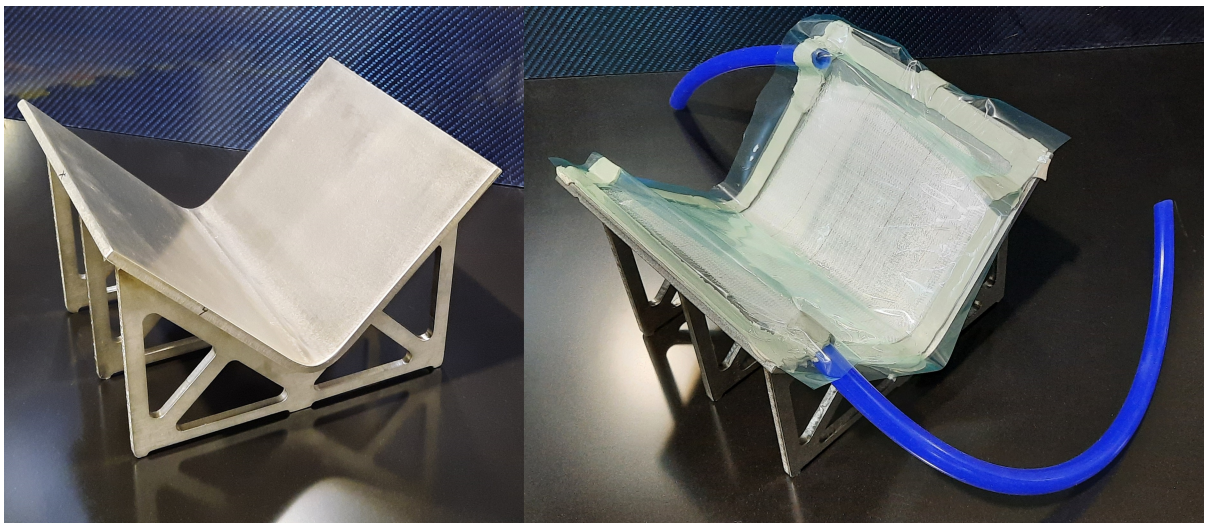


Figure 11: The steel L-shape mould used to manufacture the composite specimens before (left) and after the bagging process has been completed (right).

736 and aluminium alloys in order to monitor the effect of tool material on PID of the structures.
 737 Other factors that affect PID were also experimentally investigated such as specimen thick-
 738 ness and stacking sequence. Fig. 11 depicts one of the three moulds used to manufacture
 739 the L-shape composite specimens before and after the bagging process has been completed.

740 3.4. The global influence of local details

741 As discussed in Sec. 2.4, it is important to consider the design of some local areas during
 742 the design of the overall structure. The detail design of local areas is usually considered
 743 only after the global optimization has been performed. While this approach is not compu-
 744 tationally expensive, it does not consider in full the effect of multiple local modifications
 745 on the design of the entire structure. Therefore, there is the possibility that a full global
 746 optimization will have to be performed again, which would result in costly delays.

747 As part of the OptiMACS project, a global-local MDO approach has been developed, in
 748 order to evaluate the influence of local design parameters and to check for local constraints

749 violations already in the early stages of global design.

750 The optimization procedure is based on a monolithic architecture: the optimizer treats
 751 global and local design variables at once, thus considering the full design space.

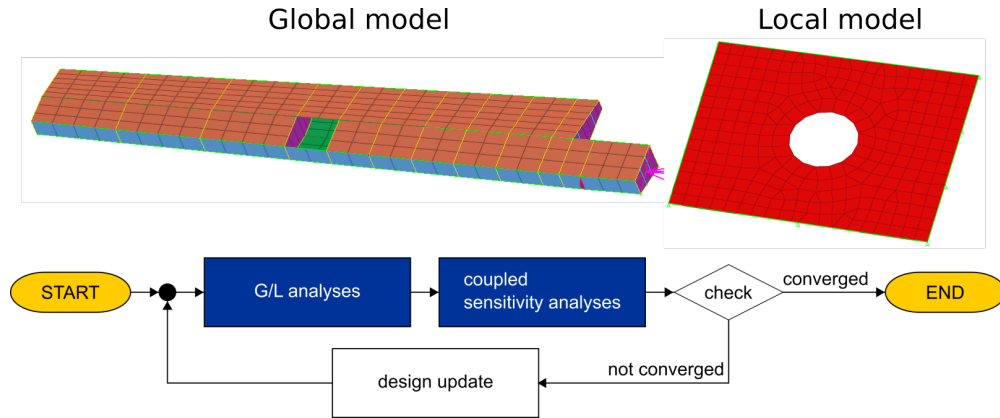


Figure 12: Global-local monolithic MDO flowchart.

752 As depicted in Fig. 12, in each optimization iteration all disciplines are solved and the
 753 sensitivities are computed, by adopting a global-local approach. In particular, two different
 754 types of analysis have been considered: linear static analysis and linear static aeroelastic
 755 analysis.

756 The global-local analysis strategy is based on three operations performed in sequence:
 757 Guyan condensation of the local information [81], solution of the global model with the
 758 condensed local information, solution of the local models. In the first step, stiffness matrix
 759 and load vector of the local models are reduced with respect to the DOFs interfaced with the
 760 global model. This results in a reduced stiffness matrix and a reduced load vector containing
 761 respectively stiffness and load contributions related to the boundary DOFs shared by the
 762 global and the local models. Next, the entries of these reduced quantities are added to the
 763 stiffness matrix and load vector of the global model. With the added local information, the
 764 resulting solutions of the global analyses are affected in such a way as to take into account
 765 the influence of the local models. In particular, in the case of static analysis and static
 766 aeroelasticity, the solutions are the same that would be obtained, if a unique model with
 767 the same mesh, obtained by joining global and local models, was solved. In the last step,
 768 the local models are separately solved by using the computed global solution as a Dirichlet
 769 boundary condition, applied at the interface between global and local models.

770 Once the analyses are completed, a coupled sensitivity analysis is performed. This global-
 771 local sensitivity analysis takes into account global and local constraints and global and local
 772 design variables. Thus, it captures the interaction between global and local design choices or,
 773 more precisely, the influence of global design variables on local constraints and the influence
 774 of local design variables on global constraints. This essentially means to capture the influence
 775 of the design variables of one model on the solution of another one. And since the solutions
 776 are computed following a special global-local analysis strategy, the global-local sensitivity
 777 analysis requires an ad-hoc formulation.

778 Thanks to the fact that the sensitivity analysis considers local constraints and design
779 variables as well, the optimizer can then take advantage of the additional freedom provided
780 by the local design space, while at the same time ensuring the feasibility of the local design.
781 Furthermore, by computing the sensitivities of active constraints only, the impact on the
782 overall computational cost is limited.

783 *3.5. Seamless integration of software tools*

784 As indicated in Sec. 2.5, there is a need to improve the integration of the software
785 packages used in the design evaluation process by automating data generation and transfer
786 between these packages. In the work carried out during the OptiMACS project, implemen-
787 tation of this automation has been focused on the structural interface between wings and
788 fuselage. Several bottlenecks in the data transfer process have been identified and addressed
789 in this work, such as i) definition of structural interfaces; ii) definition of wing cut-out;
790 iii) automated assignment of sizing variables and constraints; iv) automated processing for
791 flight conditions and load cases; and v) automated generation of aero-structural coupling
792 input. The structural interfaces are categorised as discrete and continuous [120], where for
793 the discrete structural interface, attributes are defined for joint position, stiffness, material
794 properties and thickness of the joint elements. For the continuous structural interface, refer-
795 ences for the connecting structures are created as well as a reinforcement structure such as a
796 cruciform, triform or buttstrap. Furthermore, a new cut out element, defined by ribs, spars,
797 and/or relative coordinates, has been created and used to define a patch on one side of the
798 wing skin with different material properties or stringer definition. Moreover, a method for
799 automatic assignment of sizing variables based on information recorded from choices made
800 by engineers has been developed and implemented as a Python program. Similarly, a tool for
801 automated reading of the the information from the design file and conversion of the param-
802 eters required by the optimizer has been implemented. Finally, to allow for coupling between
803 the structural and aerodynamic model, the tool for automatic generation of coupling input
804 has also been developed.

805 The developed interfaces, highlighted in blue boxes in Fig. 13, have resulted in a stream-
806 lined process giving a significant reduction of the time taken from an average of two months
807 to approximately an hour (for a large airframe design with over 10^5 DOF, about 8×10^4
808 sizing variables and 2×10^5 constraints in the FEM structural model). A large proportion
809 of the time needed for current automated evaluation is attributed to the sizing optimization
810 in *Lagrange*, followed by the generation of the structural model. With this automation,
811 it greatly improves the overall efficiency for airframe design evaluation and opens up the
812 possibility of MDO for the airframe design.

813 **4. OptiMACS contributions to the efficient and optimal design of airframe struc-** 814 **tures — case studies**

815 *4.1. More efficient stacking sequence optimization for aircraft structures*

816 The two-stage optimization process presented in 3.1 has been applied to the wing covers
817 of OptiMALE, an industrial-scale demonstrator shown in Fig. 2. The aircraft is modelled

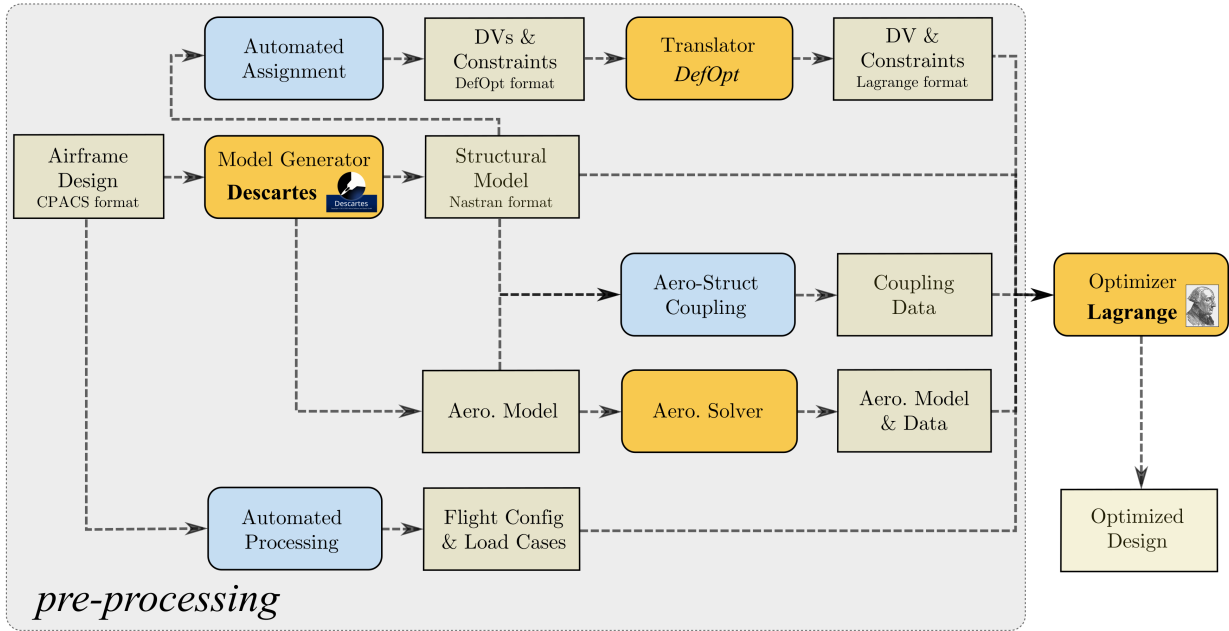


Figure 13: The improved workflow of an airframe design performance evaluation.

818 using a coarse Global FE model (GFEM) consisting of 1D and 2D structural elements. The
 819 model is subjected to 19 static load cases which have been preselected from a complete flight
 820 envelope covering different operating altitudes, Mach numbers and load factors. In this work,
 821 only the wing of the aircraft is studied and therefore the wing skins, spars, stringers and
 822 ribs are represented using a total of approximately 3000 design variables. Besides strength
 823 and buckling constraints, manufacturing constraints are also applied to the skin of the
 824 wing. More specifically, blending and maximum ply drop constraints are applied between
 825 neighbouring laminates. This leads to a total of more than 570,000 constraints.

826 The outer part of the wing of the aircraft is detachable due to storage requirements
 827 which leads to a total of 4 sub-components, 2 for each of the upper and lower parts of
 828 the wing skin. The patches on the upper and lower skin of the wing which can be seen
 829 in Figs. 14a and 14b have been chosen manually, using more, smaller patches towards the
 830 root of the wing where the thickness gradients are expected to be steeper. A total of 134
 831 patches have been used in this study. Concerning the number of generic layers used to
 832 model the stiffness properties of the structure, a maximum of 32 generic layers resulting
 833 in 12 design variables due to symmetry and balance requirements have been used for the
 834 thickest regions of the wing covers. For the thinnest, outer parts of the wing, 8 generic layers
 835 leading to 3 design variables for each patch have been chosen to model the properties of the
 836 structure. Modelling each patch with an appropriate number of generic layers depending on
 837 the expected thickness is of high importance, because an inadequate generic stack can lead
 838 to an erroneous continuous stiffness outcome, which in turn cannot be matched during the
 839 discrete optimization stage resulting to violation of physical constraints in the structure. For
 840 example, using significantly more generic layers than the actual thickness of the patch, offers

841 a large design freedom which cannot be matched in the discrete stage. On the other hand,
842 when two neighbouring patches are modelled with a different number of generic layers, the
843 blending constraint cannot be formulated precisely on a layer to layer basis but is rather
844 inexactly applied to the total number of layers per orientation [85]. This can in turn also
845 lead to continuous results that cannot be precisely matched in the second stage. Therefore,
846 the number of neighbouring patch interfaces where a different generic stack is used must also
847 be kept to a minimum. The gradient-based optimization converges to a continuous thickness
848 distribution of the skins of the wing which results to a mass of 226.8 kg.

849 The total, continuous thickness of each patch is rounded up to an integer number of dis-
850 crete layers, while maintaining the number of each individual fibre orientation $[0, 90, \pm 45]$
851 above a safety threshold to assist with the satisfaction of strength constraints in the dis-
852 cretized structure. The discrete optimization is performed using the decomposition technique
853 mentioned in Sec. 3.1. Small physical constraint violations were observed after evaluating
854 the discretized solution with *Lagrange*. For the skins of the wing, the minimum Reserve
855 Factor (RF) observed for strength was 0.99. The RF is the ratio of the allowable over the
856 applicable load, so a $RF < 1$ indicates a constraint violation. Slightly bigger violations were
857 observed for the buckling constraints of the spar webs and stringers, namely a minimum
858 RF of 0.93 and 0.90 respectively. These components were not discretized and the reason
859 for the constraint violations is load redistribution due to the discretization of the wing skin
860 laminates which attracted more loads in some areas. One solution to the constraint vio-
861 lations of the discretized structure is to increase the design factor on the Finite Element
862 Model and perform the two stages of the optimization again. However, this process is time
863 consuming and would also end up increasing the weight of the structure quite significantly
864 due to multiple components of the wing being unnecessarily overdimensioned. Instead, since
865 only very minor constraint violations were observed for the wing skins which were the parts
866 of the structure to be discretized, the first stage of the optimization was performed again
867 while keeping these discrete laminates constant during the optimization. Instead, the design
868 variables for the spar webs and stringers, which were not discretized, were all active. This
869 led to the fulfilment of all structural constraints on the wing for both the spar webs and
870 stringers, but also the wing skins. The mass penalty during this corrective process was only
871 3 kg with the discretized wing skin being 235.1 kg showing only a 3.7% increase compared
872 to the original continuous result.

873 Even though the contributions of the work performed on the stacking sequence opti-
874 mization towards the overall detailed sizing capabilities of *Lagrange* cannot be explicitly
875 quantified, they can be divided in two categories. First of all, the introduction of the second
876 stage of the optimization automates the retrieval of good quality solutions satisfying a wide
877 range of guidelines. Since this task used to involve a lot of manual effort and re-iterations,
878 the sizing process has become more time efficient. Secondly, the introduction of composite
879 constraints in the first stage of the optimization, bridges the information gap between the
880 two optimization stages leading to minor constraint violations after the discretization of the
881 structure which would otherwise need to be resolved by introducing large design factors.
882 Besides forcing the entire two-stage process to be repeated again, these design factors would
883 also lead to over-dimensioning of entire components, leading to a significantly higher mass

884 penalisation.

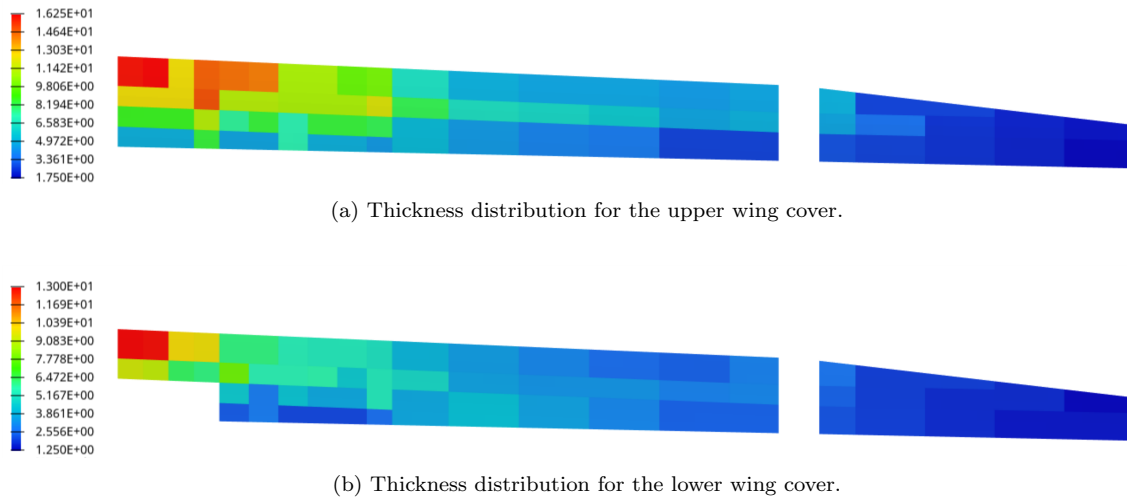


Figure 14: Thickness distribution for the final, discrete laminates of the OptiMALE demonstrator aircraft.

885 *4.2. Integration of detailed failure models within the MDO for accurate and efficient damage-*
886 *resilient aircraft design*

887 This section presents examples of application of the failure methods introduced in Sec. 3.2.
888 In the next subsection, extended omni strain failure envelopes are correlated with experi-
889 mental data from the literature in order to validate their predictions for composite laminates.
890 Then, the following subsection presents a case study of the “hot spot” identification method.

891 *4.2.1. Validation of extended omni strain failure theory*

892 The failure theory outlined in Sec. 3.2.1 was developed to generate fast and safe predic-
893 tions of failure for FRP laminates. Herein, to assess the performance of this failure theory,
894 extended omni strain failure envelopes are tested against experimental evidence. Further-
895 more, the obtained failure envelopes are compared with the omni strain failure envelope
896 based on Tsai-Wu failure theory, to study the strengths and limitations of the proposed
897 extension.

898 In particular, a validation study of the predicting capability of the extended omni strain
899 failure concept was performed using experimental results from the first and second World-
900 Wide Failure Exercise (WWFE) [121, 122]. Firstly, several test cases from the WWFE-I,
901 involving multidirectional laminates under biaxial loads, were selected. Among these test
902 cases, two are shown in Fig. 15, where omni FPF/LPF envelopes, obtained using the 3D
903 invariant-based failure theory and Tsai-Wu, are correlated against experimental data for a
904 AS4/3501-6 [90/±45/0]s laminate (Fig. 15a) and a E-glass/LY556/HT907/DY063 [±30/90]s
905 laminate (Fig. 15b).

906 An excellent agreement can be observed for the 2D test data when considering omni LPF
907 envelopes, except for the compression-compression quadrant where the predictions overes-
908 timate the laminate strength under biaxial compression. These less accurate predictions

909 are justified by a reported buckling occurred in those specimens, leading to a premature
 910 failure in both laminates [122]. The biaxial test cases provide also a clear indication on the
 911 huge benefits in using a LPF approach instead of FPF predictions. The larger domain when
 912 using LPF predictions allows to reduce conservatism in a remarkable way, without incurring
 913 additional computational time. These benefits can be exploited immediately from the con-
 914 ceptual design stage of composite aerostructures, since the presented tool is invariant with
 915 respect to the laminate layout. The beneficial impact of this approach on the composites
 916 industry, where the consolidated practice in early design stage is to use FPF theories, such
 917 as maximum strain or Tsai-Wu criteria, can be significant.

918 It can be noted that a good correlation with these experimental data was already achieved
 919 by competing failure theories involved in the WWFE (such as the criteria developed by Puck
 920 and Schürmann) [122]. However, the unique feature of the omni strain failure concept (for
 921 both theories) is that laminate failure predictions require only the material properties ex-
 922 tracted from the UD material. Despite the two omni strain failure envelopes provide similar
 923 failure prediction for biaxial test cases, the added value brought by the proposed envelopes
 924 can be still highlighted when analysing glass-fibre composites, whose LPF is governed by
 925 different failure modes (as shown in [94]); LPF of CFRP laminates, on the other hand, is
 926 always governed by fibre failure, as confirmed by this analysis.

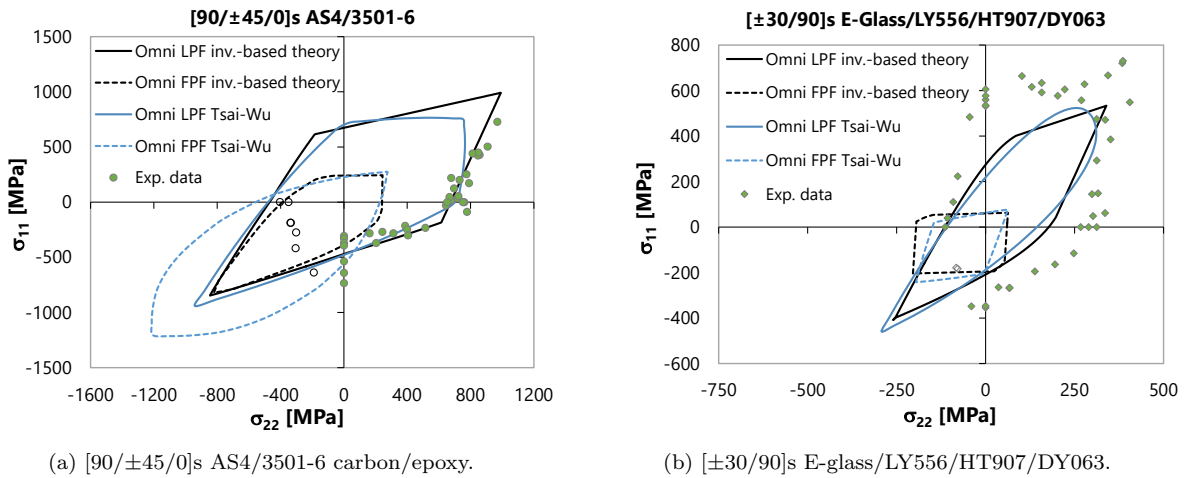


Figure 15: Omni FPF/LPF envelopes versus experimental results from WWFE-I for a $[\pm 30/90]_s$ E-glass/LY556/HT907/DY063 laminate and a $[90/\pm 45/0]_s$ AS4/3501-6 carbon/epoxy laminate.

927 Then, a triaxial test case for laminate failure from the WWFE-II was considered. The
 928 few available experimental results show the evolution of the transverse compressive strength
 929 σ_{22} with through-thickness stress σ_{33} (where $\sigma_{11}=\sigma_{33}$) of a glass/epoxy angle-ply laminate
 930 ($\pm 35^\circ$). Using the mechanical properties of E-glass/MY750/HY750/DY063 in the out-of-
 931 plane direction from [123], a 3D omni LPF surface was generated and correlated with the
 932 failure loci. Additionally, to assess the conservatism of the proposed 3D omni LPF surface,
 933 a laminate LPF surface obtained superposing only ply failure surfaces of the relevant orien-
 934 tations ($\pm 35^\circ$) and the same failure model, was included in this study. The correlation of

935 these surfaces with experimental data, presented in Fig. 16, shows that the laminate LPF
 936 envelope allows to reduce the conservatism of 3D omni LPF surfaces in the case of angle-
 937 ply laminates. However, the omni LPF envelopes define, in a physically-based setting, a
 938 safe approach for laminate failure prediction that is independent of the particular stacking
 939 sequence, thus can be applied to any laminate of a given material system. This can be
 940 better assessed in Fig. 17, where the relevant sections of these failure surfaces are compared
 941 with experimental data. In this figure, the omni strain LPF envelope based on Tsai-Wu
 942 failure theory are also included, showing that the design space is considerably reduced in
 943 the first and third quadrant, as a result of the effect of the out-of-plane stress, accounted in
 944 the omni LPF surfaces. This means that the influence of the out-of-plane stress cannot be
 945 neglected to properly capture laminate failure under hydrostatic pressure and to obtain safe
 946 LPF predictions under general 3D stress states.

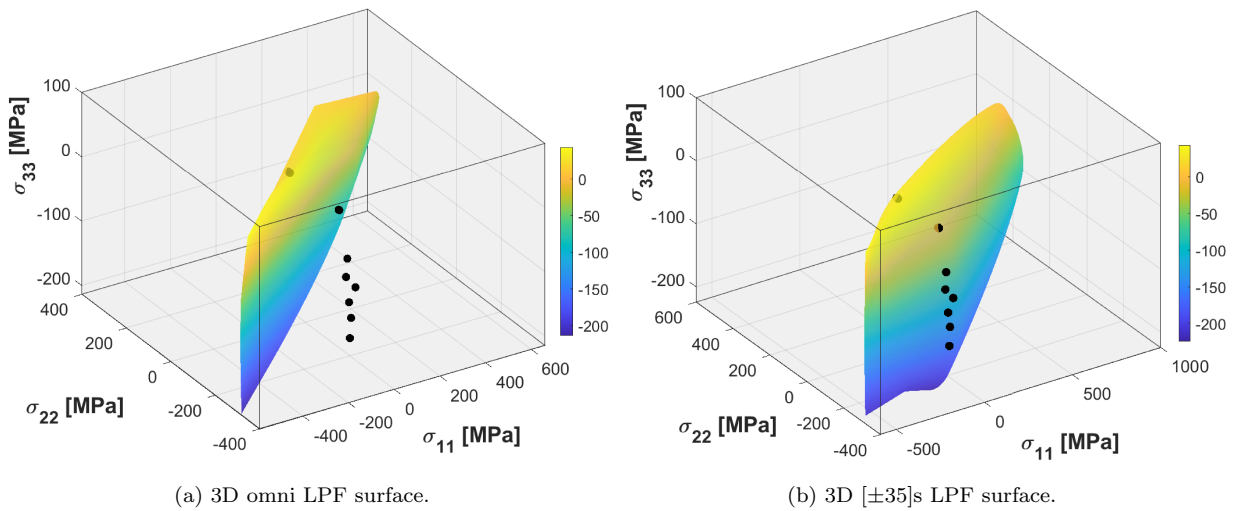


Figure 16: 3D omni LPF (a) and $[\pm 35]_s$ laminate LPF (b) surfaces versus experimental results from WWFE-II for a $[\pm 35]_s$ E-glass/MY750 epoxy laminate.

947 4.2.2. Example of application of the detailed failure model

948 An example of application of the “hot spot” identification method, as described in
 949 Sec. 3.2.2, is shown in Fig. 18. In this case, the hot-spot failure analysis is applied to an
 950 aeronautical reinforced panel, targeting the identification of the critical locations for damage
 951 onset in the runout region. For the discretization of the structure under analysis, first-order
 952 solid elements are preferred over shell elements, because only the first ones can account for
 953 components of the full set of the stress tensor, playing a crucial role in the runout region,
 954 where a change of load path takes place.

955 Since the results show that the onset of damage in the skin region close to the runout is
 956 triggered by fibre kinking, a detailed model of that area was built to perform local failure
 957 analysis and study the fibre kinking onset and broadening. In this way, the evolution of
 958 damage due to fibre kinking can be studied up to collapse. However, in general, a material

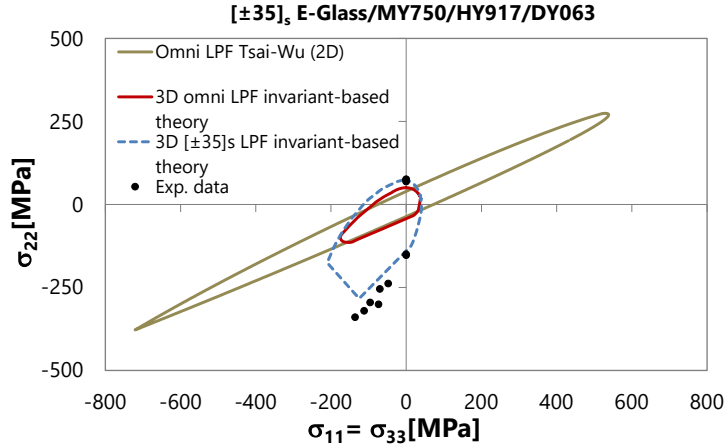


Figure 17: Correlation of 3D omni LPF, 3D laminate LPF and omni LPF based on Tsai-Wu with experimental data from WWFE-II for a $[\pm 35]_s$ E-glass/MY750 epoxy laminate.

959 model representing the initiation and propagation of all failure modes can be also imple-
 960 mented. In this way, the mechanical response of the reinforced panel up to final collapse
 961 can be predicted more accurately, but a full model involves higher computational costs.

962 Additionally, in those areas where delamination or debonding is predicted to take place,
 963 cohesive elements can be introduced in the FE-model to accurately predict the onset and
 964 propagation of these phenomena. As an example, in Fig. 19, the delamination growth in
 965 a open-hole laminate is shown by highlighting the cohesive elements with different colours:
 966 in green the ones partially damaged (with a damage variable between 0.10 and 0.99), in
 967 blue the ones where damage is in its early stage (with a damage variable between 0.001
 968 and 0.10) and in red the ones severely damaged (with a damage variable greater than 0.99).
 969 The elements that are not damaged take an initial colour which has been set as white. The
 970 damage variable of the cohesive elements is calculated from the evolution law implemented
 971 in Abaqus and proposed in [124].

972 As a remark, OptiMACS research on failure models allowed mainly to deliver two novel
 973 contributions: i) the development of a fast tool to predict laminate LPF under general 3D
 974 stress states through the concept of omni LPF envelopes and ii) an extended composite
 975 material model to account for the effect of out-of-plane stress components in the initiation
 976 and broadening of fiber kinking.

977 4.3. Manufacturing distortions

978 The CHILE and the viscoelastic material models presented in 3.3 were applied to predict
 979 PID of L-shape composite structures. A thermo-chemo-mechanical simulation approach was
 980 employed in this case and tool part interaction was also investigated regarding its effect on
 981 simulation results. By performing an extensive numerical investigation on these structures
 982 it was found that the fixed boundary condition produces the minimum distortion (spring-in
 983 angle), whereas the free-standing boundary conditions the maximum expected distortions
 984 (spring-in angle). The tool part interaction predictions lie in between the values predicted

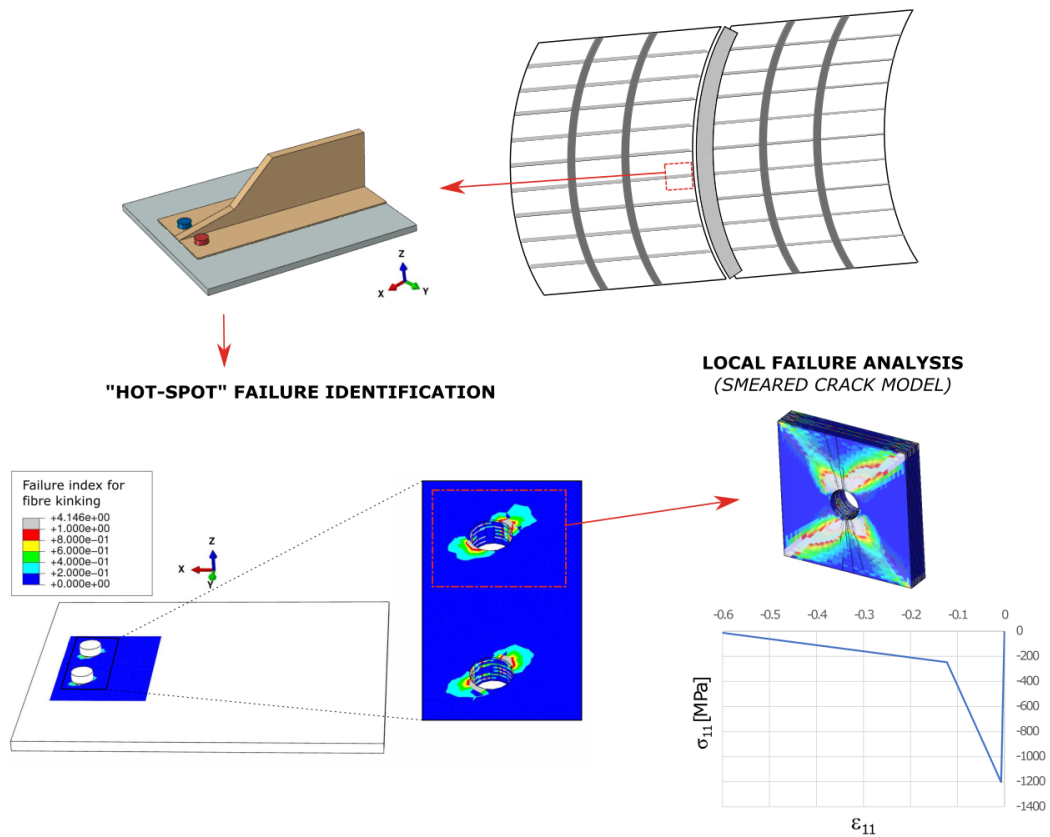


Figure 18: Illustration of the local analysis strategy to capture failure with detailed damage models.

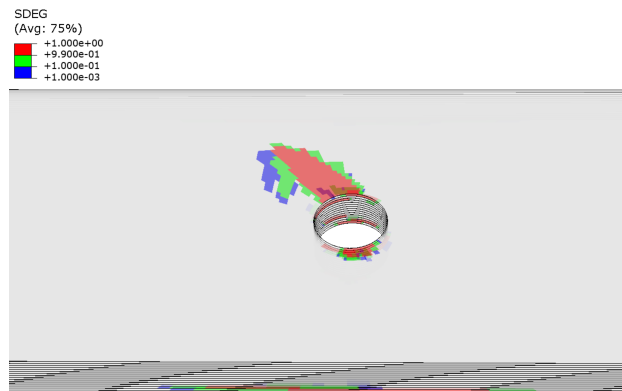


Figure 19: Prediction of delamination growth for a quasi-isotropic open-hole laminate, by using a damage variable for cohesive elements (transparency level: 50%).

985 by the fixed and free-standing boundary conditions for most of the cases studied. Fig. 20
 986 depicts a comparison of the distortion predicted by employing different boundary conditions
 987 for the case of an L-shape structure having a stacking sequence representative of an aerospace
 988 frame.

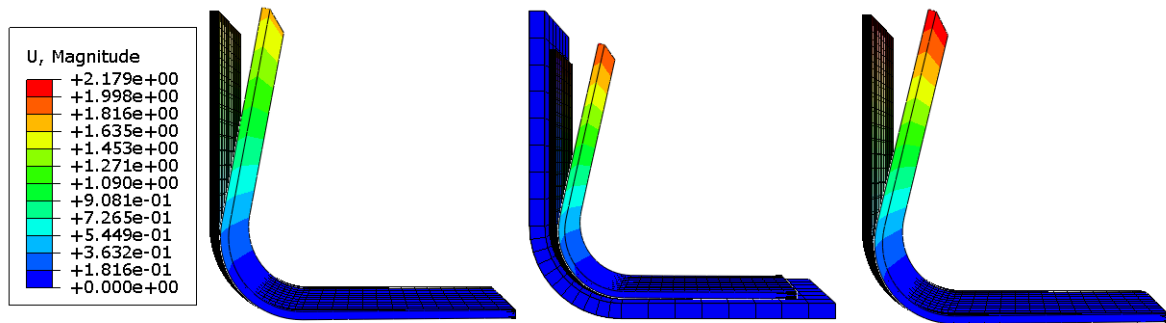


Figure 20: Distortions in mm of a L-shape composite structure at the end of the curing cycle using the CHILE material model with the use of fixed (left), tool-part interaction (middle) and free-standing (right) boundary conditions.

989 The CHILE and the viscoelastic material models were also applied to predict PID of
 990 an aerospace test frame of industrial size (Fig. 21). Even if this frame is only intended for
 991 research purposes, it has many common features with flying frames used in aircraft fuselages
 992 as the one depicted in Fig. 2 for the case of OptiMALE. To simplify the analysis, in this case
 993 study a chemo-mechanical simulation approach was adopted by assuming a homogeneous
 994 temperature field across the part at every time step, which is a product of manufacturing
 995 experience.

996 By comparing the predictions of the two material modes with the 3D scanned shape of
 997 the frame, it was found that the viscoelastic material model could predict more accurately
 998 the shape distortion of the part compared to the CHILE material model which was found
 999 to overestimate in magnitude the distortion of the frame [70, 125]. Consequently, in the
 1000 context of the OptiMACS project, viscoelastic material models are proposed to predict
 1001 shape distortions of aerospace thermoset composite parts when the maximum prediction
 1002 accuracy is sought. However, taken into account the increased material characterization
 1003 effort and cost needed by viscoelastic material models to run, along with their increased
 1004 calculation time due to the calculation and storage in memory of state variables, the CHILE
 1005 material model is regarded to be a good compromise between cost and performance.

1006 Regardless of the material model chosen to predict PID of the frame (CHILE or vis-
 1007 coelastic), the use of the simulation framework developed in OptiMACS (Fig. 9), enabled
 1008 the prediction of a complex distortion field (Fig. 21). This could not be predicted by simple
 1009 analytical equations or manufacturing experience, usually employed in the shop floor.

1010 Finally, Fig. 22 depicts a tool part interaction simulation that was performed by em-
 1011 ploying an aluminium mould and a cure dependent CoF of a U-shape composite structure.
 1012 It was found that the aluminium mould compresses the composite part at the end of the
 1013 cool down phase (end of curing cycle) due to the great difference of CTE of the aluminium
 1014 and composite structure, showing that tool part interaction plays an important role in this
 1015 application.

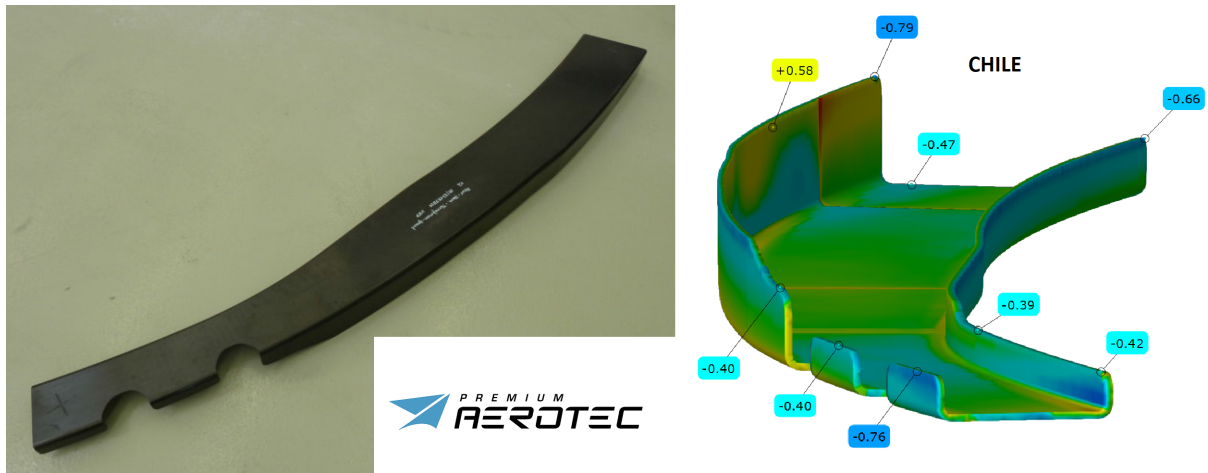


Figure 21: Industrial test frame (left). The expected distortions of the frame with the use of the CHILE material model (right).

1016 4.4. Global-local MDO

1017 The procedure for global-local MDO presented in Sec. 3.4 can be applied to the weight
 1018 minimization of a model like the wingbox represented in Fig. 23.

1019 The rectangular area highlighted in red represents a local structure modelled in a separate
 1020 FE-model. The entire structure is optimized by modifying 159 design variables, representing
 1021 the thicknesses of shell elements and cross sectional area of bar elements. Of these 159
 1022 design variables, one is used to design the local model while instead 158 parametrize the
 1023 global model. Strength constraints are applied to both the global and the local model. By
 1024 applying the monolithic approach presented in Sec. 3.4, it is possible to obtain the optimized
 1025 thickness distribution of the structure, while satisfying both global and local constraints.

1026 Fig. 24 shows a comparison between a reference thickness distribution and the global-
 1027 local optimal thickness distribution obtained for the same static analysis subcase. Fig. 24a
 1028 shows the optimal thickness distribution obtained without the application of a global-local
 1029 strategy and using a single coarse model, which does not capture in detail the local geometry.
 1030 Using a separate refined local model and the global-local strategy presented in Sec. 3.4, the
 1031 obtained optimal thickness distribution is the one shown in Fig. 24b.

1032 Analogously, Fig. 25 shows the same comparison for an aeroelastic analysis subcase.

1033 The global-local analysis of each subcase is solved by condensing the local model, solving
 1034 the global model by adding the local contributions and solving the local one with the global
 1035 solution as a boundary condition. The sensitivities are computed with the global-local
 1036 methodology described in Sec. 3.4.

1037 In both cases, the obtained thickness distributions are different and, in particular, the
 1038 optimal local model design is thicker in order to satisfy the local constraints, while accom-
 1039 modating the cut-out.

1040 The reference approach was based on a fixed and unconstrained coarse representation of
 1041 the local geometry, in order to contain the computational cost of the procedure, and was
 1042 not guaranteed to yield a locally feasible final design. In contrast, the presented global-local

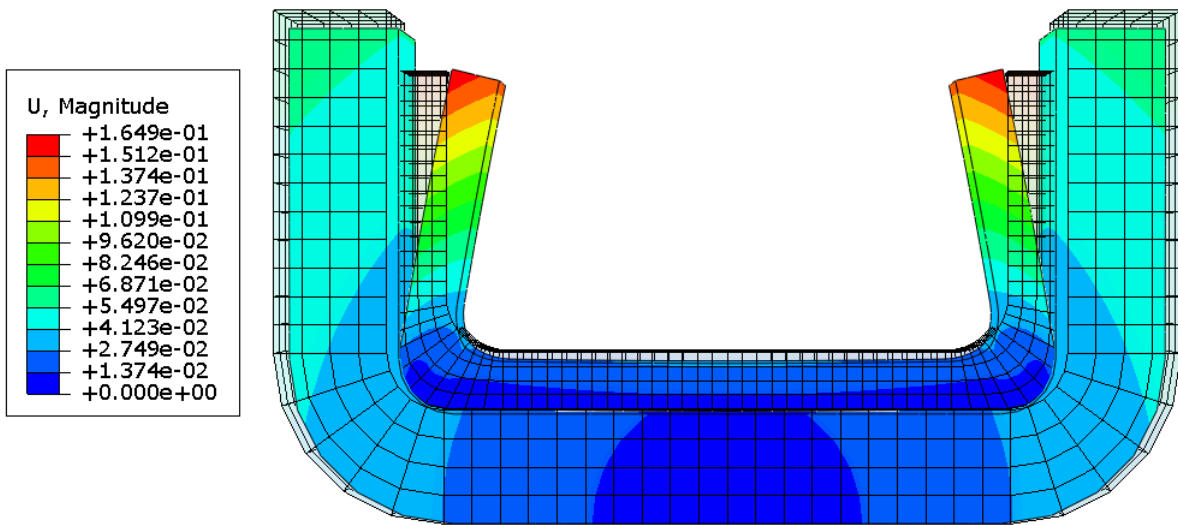


Figure 22: Distortions of a U-shape composite structure at the end of the curing cycle using the CHILE material model (cm).

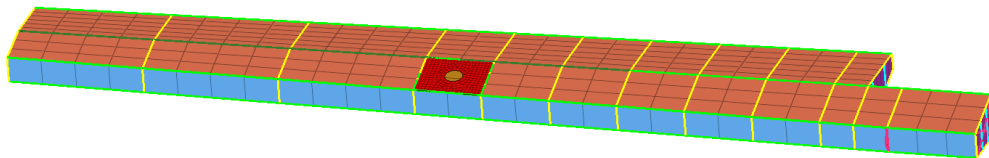
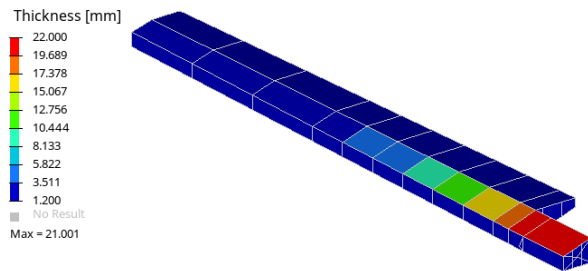
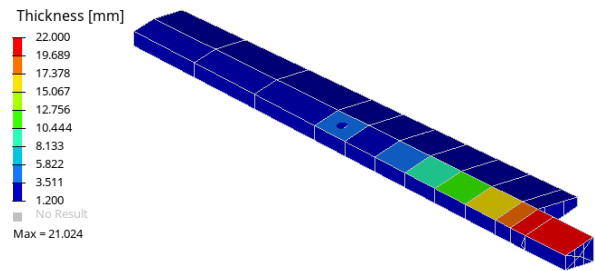


Figure 23: Global-local modelling of a wingbox.

1043 approach effectively minimizes the structural weight, while ensuring that all constraints are
 1044 not violated, including those defined over the locally refined model. Thus, the approach
 1045 effectively minimizes the chance that an update of global design will be needed.

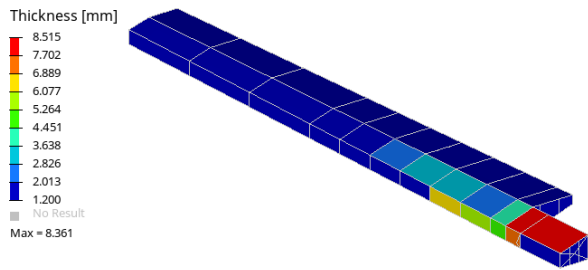


(a) Reference thickness distribution.

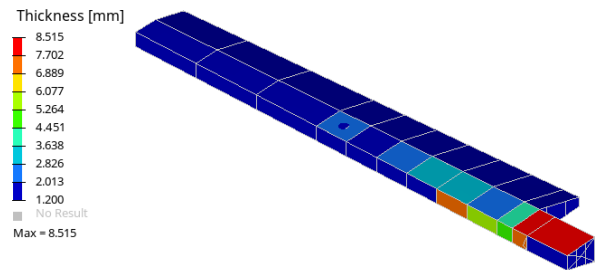


(b) Global-local optimal thickness distribution.

Figure 24: Comparison of optimal thickness distributions for a static analysis subcase: without (left) and with (right) application of local refinement and global-local strategy.



(a) Reference thickness distribution.



(b) Global-local optimal thickness distribution.

Figure 25: Comparison of optimal thickness distributions for an aeroelastic analysis subcase: without (left) and with (right) application of local refinement and global-local strategy.

1046 5. Concluding remarks and anticipated future developments

1047 Some of the major industrial challenges to achieve optimal structural designs are hereby
1048 discussed. The need for accurately capturing the performance characteristics of each design
1049 candidate early in the preliminary, or even in the conceptual stage has provided motivation
1050 and impulse towards new MDO technologies. Emphasis is particularly given on the op-
1051 portunities and challenges arising through the employment of composite materials. Recent
1052 developments in the aeronautical sector are globally oriented towards capturing more detail
1053 of the structural product within the MDO loop at minimum or no additional computational
1054 cost.

1055 In this manuscript we discuss the challenges and potential impact of the following specific
1056 structural disciplines which are under intense development within the modern aeronautical
1057 industry. The contribution of the OptiMACS project towards these contemporary challenges
1058 is also exhibited with concrete case studies also provided.

- 1059 • The optimal design of large laminated structures with intense thickness variations over
1060 their surface is associated with a number of challenges due to the mixed discrete and
1061 continuous nature of the problem. Hence, in the place of a traditional design develop-
1062 ment process, two-stage optimization approaches are becoming more popular within
1063 the aeronautical sector. Such approaches are hereby discussed. It is demonstrated
1064 that a two-stage optimization process has the potential to lead to solutions satisfying
1065 all required structural constraints for a lower component mass.
- 1066 • Accounting for structural resilience against damage accumulation early in the design
1067 process is another major challenge for the next generation of MDO processes. Inclusion
1068 of more accurate failure criteria is expected to enable lighter designs through relaxation
1069 of safety factors as well as inclusion of phenomena such as reversible local buckling. The
1070 manuscript discusses the application challenges and development of global-local failure
1071 methodologies. As a case-study representative for the aeronautical sector, extended
1072 omni-strain failure envelopes are correlated with experimental data from the literature
1073 in order to validate their predictions for composite multi-directional laminates, while
1074 the hot-spot failure identification method is employed to predict the most critical
1075 areas and failure modes in a reinforced panel, addressing then the critical regions with
1076 detailed damage models.
- 1077 • Addressing the presence of process induced distortions is another major challenge when
1078 composite materials are to be implemented in the design. This is mainly due to their
1079 peculiarities related to resin chemical shrinkage, tool part interaction, temperature
1080 gradients and stress relaxation amongst other factors. The computational challenge of
1081 considering these manufacturing parameters in the design process of moulds, having a
1082 optimized shape, in order to achieve a “First Time Right” approach in the manufac-
1083 turing of composite structures, was also discussed in the manuscript.
- 1084 • The integration of local structural complexities within the MDO procedure generally
1085 comes with added computational burden. It is however critical for accounting the

1086 impact of such complexities on the airframe design and avoiding an extremely conser-
1087 vative, sub-optimal design due to lack of this local information. In this manuscript a
1088 global-local analysis strategy is discussed, based on Guyan condensation of the local
1089 information and subsequent solution of the global and local models. A global-local sen-
1090 sitivity analysis, combined with an active set optimization strategy, allows to account
1091 for local constraint violations at an acceptable computational cost.

- 1092 • The seamless integration of software tools is another major challenge especially when
1093 an integrated multiscale framework is to be implemented able to exchange informa-
1094 tion between the preliminary and conceptual design stages. We hereby discussed chal-
1095 lenges related to structural interface definitions, assignment of sizing variables and
1096 constraints, automated processing for flight conditions and load cases, as well as au-
1097 tomation of the aero-structural coupling.

1098 Future work in the aeronautical MDO sector is expected to rely on an increased level of
1099 detail during preliminary sizing of a structural model and complete quantitative evaluation
1100 of its performance with a limited computational cost. For instance, research is currently
1101 working to account for inspectability and manufacturability aspects in the MDO process,
1102 which is found to play a crucial role in the lifecycle of an aircraft. Pushing MDO within
1103 the conceptual design stage will be challenging for structural engineers over the next few
1104 years; it is however certainly the global vision over decades to come. Such an advancement
1105 will effectively *erase any solid boundaries between the conceptual and preliminary stages,*
1106 *eventually unifying the design optimization process.* Multidisciplinary developments in the
1107 fronts of more efficient physics models, metamodelling (efficient surrogate predictions in or-
1108 der to radically reduce design evaluation times), parallel computing and FE model reduction
1109 schemes are all very welcome in order to synergistically achieve the above vision.

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1115 List of abbreviations

1116 **ATL** Automatic Tape Laying

1117 **CFRP** Carbon Fibre Reinforced Plastics

1118 **CHILE** Cure Hardening Instantaneous Linear Elastic

1119 **CMM** Coordinate-Measuring Machine

1120 **CoF** Coefficient of Friction

1121 **CPACS** Common Parametric Aircraft Configuration Schema

1122 **CTE** Coefficient of Thermal Expansion

1123 **DLR** Deutsches Zentrum für Luft- und Raumfahrt - German Aerospace Center

1124 **DOF** Degree Of Freedom

1125 **FE** Finite Element

1126 **FEM** Finite Element Method

1127 **FPF** First-ply failure

1128 **FRP** Fibre-reinforced polymer

1129 **GFEM** Global FE model

1130 **GUI** Graphical user interface

1131 **LPF** Last-ply failure

1132 **MDO** Multidisciplinary Design Optimization

1133 **MILP** Mixed Integer Linear Programming

1134 **NCF** Non-Crimp Fabric

1135 **OptiMACS** Optimization of Multifunctional Aerospace Composite Structures

1136 **PID** Process Induced Distortions

1137 **RBDO** Reliability-Based Design Optimization

1138 **RDO** Robust Design Optimization

1139 **RF** Reserve Factor

1140 **TIGL** TIVA Geometric Library

1141 **TIVA** Technology Integration for the Virtual Aircraft

1142 **UD** Unidirectional

1143 **XML** eXtensible Markup Language

1144 **XSD** XML Schema Definition

1145 **WWFE** World-Wide Failure Exercise

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